

STEEL SHIELD TECHNOLOGIES HVAC System

We're the Champion, let's make yours !!!



www.steelshieldtech.com.hk Serving the Industry since 1985

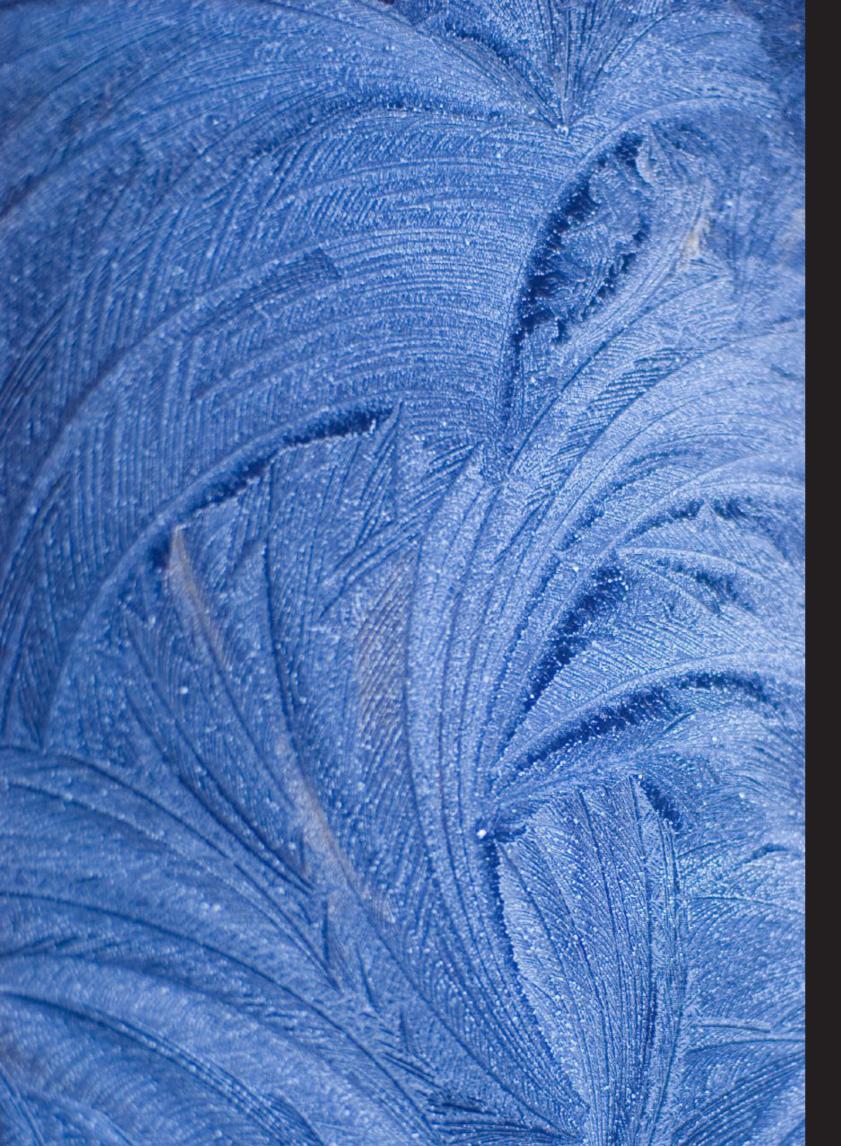


World's 1st Ionic-Maglev Lubrication Technology

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WORLD'S IST IONIC-MAGLEV LUBRICATION TECHNOLOGY

I. MAKING A DIFFERENCE IN LUBRICATION

"It is our conviction that to be the best is not sufficient, we are here to make a new World Standard in Lubrication."

Company Vision & Commitment

Steel Shield Technologies sole purpose is to manufacture premier quality metal treatments, additives, greases and lubricant oils that have been tested to exceed the normal parameters of extreme pressure and antiwear products in the aftermarket, hereby offering matchless performance and unsurpassed protection against wear while saving maintenance costs, downtime, energy and improving overall functionality of your machineries.

Steel Shield, Not Just Oil, It's Technology which makes a difference to the World of Lubrication.

Steel Shield aims at helping customers to achieve the highest return on investment (ROI). Steel Shield is committed to strengthening business and global commerce through manufacturing and distributing, World-wide, the full line of ABF Technology products made in the USA, Singapore and Hong Kong.



THE CORPORATION & FACILITIES 2.

Steel Shield Technologies Inc. (USA) with its history traced back to 1985 when in USA, Pennsylvania the scientist Dr. George C Fennell in the research and development of high-end specialty lubricants invented the unique ABF Formula - Ionic Levitation. In the same year Muscle Product Corporation trading as MPC was founded by George Fennell, brother Jay Fennell and father, Richard Fennell and the purpose was to market his invention MT-10. In 2006 at a board meeting held on 22nd May George resigned all his duty from MPC. In the same year George found STEEL SHIELD TECHNOLOGIES INC and renamed his invention MT-10 as Steel Shield. As of then MPC and its products are no longer being supported by George either in performance or quality.

The Company's blending and manufacturing capabilities are state of the art and the ability to produce limitless volume of product is unsurpassed as well as the product quality. The equipment is all stainless steel including the flow lines, pipes and couplers. All pumps and gauges are digitally interpreted and of the highest quality and accuracy to ensure production of the most superior quality lubricants.

STEEL SHIELD TECHNOLOGIES (ASIA PACIFIC) LIMITED WAS INCORPORATED IN 1996 IN HONG KONG TO PROVIDE DISTRIBUTION AND TECHNICAL SUPPORT FOR THE ENTIRE ASIA-PACIFIC RIM.



3. INVENTOR SCIENTIST -DR. GEORGE C FENNELL

Accreditation:

STLE



In 1986, Dr. George C Fennell, a former scientist in Astronomy and Astrophysics doing consulting and contract work in advanced lubrication and surface Tribology, formulated a revolutionary metal treatment oil additive which can activate "ABF" (Advanced Boundary Film) through a proprietary and unique "electrochemical ionization" (ECI) process. He has been known in the industry as the "Father of ABF Lubrication".

On the basis of ABF technology, a series of specialty lubricants have been developed to meet the stringent requirements of various purposes and working conditions, as to date is still the most advanced formula in lubrication.

Over the years, there have been countless people trying to resemble Dr. Fennell's unique formula and advanced chemistries, none was found even remotely close. To this date, Dr. Fennell is still the leader in tribology and lubrication.







Father of ABF Technology **Doctor of Astronomy and Astrophysics**

- SAE (Society of Automotive and Aerospace Engineers) Member
- ASNE (American Society of Naval Engineers) Member
- NCMA (National Contract Management Association) Member
- STLE (Society of Tribologists and Lubricant Engineers) Member







BIO-ORGANO LUBRICATION TECHNOLOGY

During World War II, the German Science and Technology Research Institute was commissioned to develop a new lubricant technology in meeting the stringent demand for heavy duty military application such as artillery, armored vehicles, tanks, battleships and fighter-aircrafts to avail them in performing their maximum fighting capacity with minimal maintenance.

The scientists proposed the concept of Zero friction, i.e. Farady's Law Like-Charge-Repel.

Shortly after WW II, a great number of intelligent scientists migrated to the United States from Europe. One of them was the grandfather of Dr. George Fennell, who came to USA along with him a large volume of research data and material about Zero friction. The old scientist continued to pursue his scientific research and eventually in 1986 his grandson Dr. George Fennell came with a breakthrough in the technology. Through Electro Chemical Ionization (also known as Reactive Chemical Bonding) Dr. Fennell was able to realize Maglev between two metallic surfaces and to achieve a close to Zero Friction Coefficients.

The great accomplishment was the result of relentless efforts of 3 generations scientists over half a century. In recognition of the excellent contribution of Fennell's family to the country, the US Government has named the street outside their old factory Fennell Avenue as a compliment.

SOMEWHERE, SOMETHING INCREDIBLE IS WAITING TO BE KNOWN. UPERIOR

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-Carl Sagan



5. ABOUT ABF TECHNOLOGY

BOUNDARY FILM LUBRICATION THROUGH ADVANCED HALOGENATION TECHNIQUES: OXIRANE ACID SCAVENGING AND ORGANO-METALLIC SUBSTITUTION BY GEORGE C. FENNELL

Steel Shield Technologies' mechanism of operation is based upon Tribology methods that improve lubricity and load carrying capacity by improving surface characteristics and creating a stable chemical, corrosion controlled halide-based boundary film. Steel Shield's active components react with each other and the contacting asperities of the metallic surfaces to provide five mechanisms of improvement.

- 1. Advanced chemical boundary film formation through reactive chemical bonding.
- 2. Ring opening, oxirane acid scavenging and advanced corrosion inhibition.
- 3. Organo-metallic substitution of surface metal and free radical reactionaries.
- 4. Improved surface smoothness and rolling out of irregular contacting asperities.
- 5. Re-conditioning and molecular reconstruction of the original contacting metal surfaces.

The process of advanced boundary film formation is accomplished with an advanced combination of halogens that are controlled and rendered non-corrosive to the base metals of the system and pose no threat to the ozone layer or waste oil recovery systems due to their origins and long chain molecular lengths. These halogens initially react under thermal conditions with the organo-metallic reagents to form surface attaching compounds, thereby limiting and controlling the formation of halides from the base metals themselves. These surface attaching reagents or "electro-negative compounds" seek out and affix themselves to the lower surface areas referred to as micro-pores and fissures, as all metals are crystalline in structure and exhibit a lattice type matrix. This complex process also incorporates Van der Waal forces and dipole-dipole surface reactions. During this process, surface lapping and asperity (irregular microscopic contacting and opposing surfaces) roll-out is also achieved, yielding improved spread characteristics of the surfaces themselves. Due to the increase of film strength by the filling of the micro-pores and fissures, along with thermal modification of the asperities, the resulting effect is a gradual rolling out or flattening of the metal asperities rather than a breaking off or chipaway process, which would create metallic debris in the lubricant leading to abrasive wear from wear metal particles. The resulting improvement in the opposing metal surfaces further increases the fluid film strength, which is dependent on the degree of surface roughness and viscosity.

Viscosity, however, is a lesser consideration when incorporating boundary additives or halogenation techniques.

In general, boundary friction and wear consists of two components, a shear or adhesion component and a plowing or deformation component. Considering the following equation:

Fs = SAr

Where Fs is the shear component, which predominates except when asperities sink too deeply into a boundary lubricant film or a soft opposing surface. When movement or sliding occurs, the shear friction force depends on the shear resistance per unit area, S, of any "boundary film" in the real load-supporting area between asperities. Dividing by the load, W gives the shear contribution to the friction coefficient, becoming independent of total load and apparent area of contact:

fs = S * Ar / W = S / Pp or S / Pe

The boundary film shear resistance, S, is assumed equal to the plastic flow shear stress, Tp, of an ideal elastic, plastic solid. Such a solid gives shear stress independent of strain and strain rate at strains sufficiently large enough to cause plastic flow. The conditions that produce the "glass transition" from liquid to plastic-like behavior are dependent on the viscosity of the material at normal

temperatures and pressures and the variation of viscosity with temperature and pressure. In other words, glass transition depends strongly on chemical composition.

These results show that liquid lubricants act like plastic solids in the films between asperities. Therefore, S=Tp in the previous equation and the friction coefficient is Tp/Pp or Tp/Pe. Since Tp is a weak function of temperature and pressure, and Pp or Pe are independent of apparent contact load, the frictional coefficient for a given combination of lubricant and sliding surfaces tends to be independent of operating conditions.

Elasto-hydrodynamic lubrication (ELH) on an asperity scale deposits film material between sliding surfaces in "micro-rheodynamic" (micro-RHD) lubrication. As one surface slides, each asperity carries with it an aggregation of SST additive. Sufficient pressure and temperature is developed within the film to elastically deform the asperity and to force the extreme pressure reagent between the surfaces or into the micro-pores and fissures. During this time, high thermal conditions involving pressure and asperity contacts initiate a re-conditioning of the surfaces utilizing the existing oil to quench and cool the surfaces in the same process. A thermal restructuring of these asperity contact areas creates a deviation from the normal crystalline structure of the metal, expanding it into an austenitic crystalline pattern, which is more evenly structured and allows the SST additive to bond to the actual lattice of the metal, endowing it with new and unique properties upon cooling.

Organo-metallic substitution is a technique developed and designed to inhibit the process of halide formation from the base metals of the system under reaction. For example, instead of the halogen reacting with the iron in the system to form iron halides, a boundary surface salt, it reacts with a reagent having very similar properties to the iron atom itself, thereby forming a organo-metallic complex without scavenging the target metal surface itself, and depleting the metal in a chemically corrosive wear syndrome.

The process is very similar or analogous to the saponification of organo-metallic compounds in the manufacturing of greases. During this reaction or saponification, compounds react at a certain catalytic temperature and exchange characteristic components to form new compounds. These new chemical compounds are then used to aid in a boundary regime by providing an added protection to the actual surfaces being lubricated. Ring opening oxirane acid scavenging and corrosion inhibition is another chemical technique used to neutralize acids and inhibit oxidation and corrosion. This technique involves the use of specifically engineered complex ethylene oxide; oxirane rings, that possess reactive reagents which will cause a cleavage of the ring when encountering acids or strong alkaline. These reactions occur in the presence of both anionic- and cationic-type catalysts. Anionic catalysts can include alkoxide ions, hydroxides, metal oxides, and some organo-metallic derivatives while Lewis acids and protonic reagents initiate cationic reactions

The lubricity, load carrying capacity, surface improvement, and wear reduction are greatly improved while corrosive aspects of halogenation are virtually eliminated.

References:

CRC "Handbook Of Lubrication, Theory And Practice", Volumes 1 & 2, by E. Richard Booser, Ph.D., Society of Tribologists and Lubrication Engineers (STLE), copyright 1992, Eighth Printing. "Organic Chemistry" 4th Edition, by Robert Morrison, Ph.D. and Robert Boyd, Ph.D., copyright 1983 by Allen & Bacon. "Lubrication - A Tribology Handbook", edited by M.J. Neale OBE, BSc(Eng), published by Society of Automotive Engineers (SAE), copyright 1993, Butterworth-Heinemann, Ltd.

CRC "Handbook Of Chemistry and Physics", 1986 Edition, by CRC Press, edited by David R. Lide, copyright 1986 by CRC Press.

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6. HOW ABF WORKS?

teel Shield Technolog

ebster's Dictionary defines lubricants as substances capable of reducing friction, heat and wear when introduced between two solid surfaces. From the initial development and use of lubricants, chemical technology has constantly advanced to make them more effective. From changes in refinement processes to the development of additives, the concentration has always been to increase the ability of the lubricant to reduce the friction, heat and wear. Steel Shield Technologies has changed the approach to lubrication and, in essence, given new definition to the term. First, there are a few points to consider.

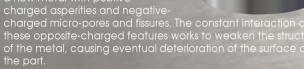
Metal Against Metal

The structure of all metals creates a surface characterized by a series of sharp peaks and valleys, some microscopic and some larger. As two metal surfaces contact each other and move in opposite directions, friction is caused, producing heat

ILLUSTRATION A

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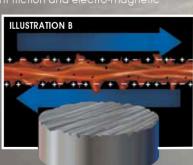
and metal deterioration. This friction-causing physical dynamic is heightened by the electromagnetic fic created on the surface known as asperifies, and valleys, referred to as m pores and fissures, have pposite electro-magnet charges. **Illustration A** sho



Normal Lubricants Help

All lubricants help to slow this process to different degrees. **Illustration B** shows the results after a period of time of use of a typical oil lubricant. The constant friction and electro-magnetic interaction has caused the weakened metal to break **ILLUSTRATION B**

off or chip away creating metallic debris in the lubric leading to abrasive wear from wear metal particles. This fact is evidenced in the need to change the engine oil of automobiles frequently as the lubricant "breaks down" due to the heat and metallic debris

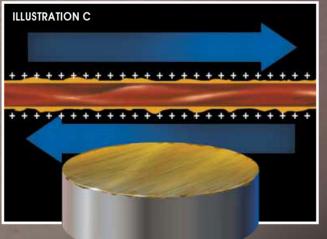


Advanced Roundary UNTREATED Wear On Unprotected Bearing TREATED ABF Technology Protects From Wear ection s has been explained, the Advanced Boundary Film Technology is a redefining approach to lubrication which provides outstanding benefits to the user.

Steel Shield Technologies has redefined lubrication by breaking away from the standard approach to making the lubricant more effective through adjusting the refinement process or through the use of additives. Instead, Steel Shield Technologies approaches lubrication by improving the surface characteristics of the metal through the process of Advanced Boundary Film formation. This technological breakthrough is accomplished by addressing the naturally formed asperities, micro-pores and fissures and the electro-magnetic charges they create.

accomplianed by dathessing the inducting formed asperities, micro-pores and fissures and the electro-magnetic charges they create. Steel Shield products consist of an advanced combination of halogens which react under thermal (heated) conditions to form electro-negative surface attaching compounds. They seek out and affix them-selves to the lower surface areas, filling the micro-pores and fissures. As this process is working, the thermal conditions are effecting the asperities. Instead of breaking off because of a weakened metal state, the asperities gradually roll out or flatten. So while the micro-pores and fissures are filling up, the asperities are flattening for an end result of a metal surface that is greatly improved. Created in this process is a total positive state of polarity. When the metal surface polarity becomes uniform in charge, there is a reduction in friction due to the Faraday reaction of like-charges. This electrochemical process continues at the molecular level to form an Advanced Boundary Film on the surface of the metal. Illustration C shows the end result of the production of the Advanced Boundary Film and the resulting uniform positive polarity.

Another aspect of this advanced technology is the ano-metallic substitution which is the chemical process designed to inhibit halide formation. Here, the



halogens used to form the surface attaching compounds react with reagents having similar properties to the iron atom. The halogens, therefore, do not scavenge the target metal surface to find iron with which to react, forming halides and creating a chemically corrosive wear syndrome. Instead, an organometallic complex is formed as the basis of the Advanced Boundary Film.

Industrial Success Comes To The Consumer

Steel Shield Technologies is now bringing this breakthrough technology to the consumer after great success on the industrial level. The level of commitment to the Steel Shield product in the railroad industry is an indication of its performance in the most extreme conditions imaginable. This same technology is now available to you.

Practical Elimination Of Metal-To-Metal Wear

Steel Shield Technologies addresses the three areas that cause the weakening and deterioration of the metal surfaces:

- The physical friction of rough surfaces
 The opposite electro-magnetic charges that exist on the metal surface
- The chemical reactions that produce
- corrosive agents. Advanced Boundary Film Technology instead strengthens

the metal and practically puts an end to metallic debris in the lubricant

Reduced Operating Temperatures

Friction is reduced so significantly that the operating temperature in treated mechanisms is notably reduced. The end result is a stronger metal that maintains its original specifications and performance level. An example of the reduction of operating temperatures is found in the independent tests that show a drop of an average of 30 Fahrenheit degrees in treated automobile engines.

Increased Effectiveness Of The Lubricant

Whatever lubricant is used as the carrier of the **Steel Shield** Technologies additive, that lubricant is allowed to perform at its maximum efficiency. Lubricant flow will be enhanced with the elimination of rough metal

surfaces; the reduction of heat and elimination of metal debris will protect the lubricant from "break down."





I. VIRTUAL ZERO FRICTION - RCB IONIC LEVITATION Faraday's Law like-charges Repel & Dipole-Dipole Reaction

2. DYNAMIC HEAT TRANSFER

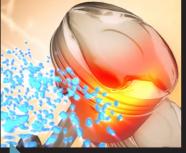
Lubricant accumulates at the hot spot automatically

3. NON CORROSIVE CLEANSING

Metal sludge repelled via induction and removed

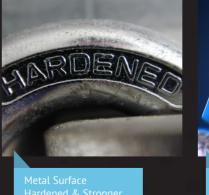






4. METAL SURFACE RE-HARDENING From Shear Friction to Surface Lapping

5. ELIMINATE SYSTEM DYSFUNCTION Not Just Oil, It's Technology









9. ABF TECHNOLOGY DETAILED EXPLANATIONS

I. METAL SURFACE

Under microscope, metal surface characterized by series of peaks and valleys, Peaks (known as "asperities"), and Valleys (referred to as "micropores" and "fissures").

2. CHARGES OF THE METAL SURFACE

Peaks are positively charged and Valleys are negatively charged.

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lpper metal bar is rotating anti-clockwi

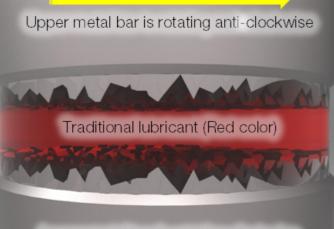
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3. TRADITIONAL LUBRICANTS

A. Traditional lubricants help slow the process of heat and friction to some degree.

B. When 2 metal surface contact each other and move in opposite directions, friction is caused, producing heat & metal deterioration.

C. Constant friction & electromagnetic interaction causes the weakened metal to break off creating metallic debris & particles in the lubricants



Lower metal bar is rotating clockwise

4. STEEL SHIELD TECHNOLOGY

A. Forms electro-negative surface attaching compounds to seek out & affix themselves to lower surface areas filling the micro-pores & fissures.

B. Asperities roll out or flatten creating greatly improved metal surfaces.

C. Created in this process is a total positive state of polarity.

D. When metal surface become uniform in charge, there is a reduction in friction due to Faraday reaction of like-charges.

5. ADVANCED BOUNDARY FILM

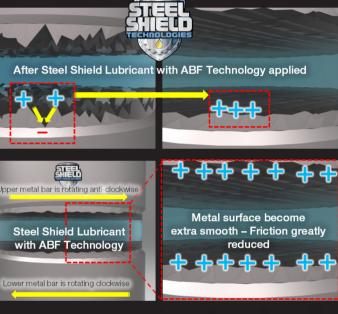
A. Advanced methods of tribology that improve lubricity and load carrying capacity

B. Reacts chemically under thermal conditions with the contacting metal surfaces to form a complex surface-attaching film of protection

C. Surface smoothing is accomplished resulting in improved spread characteristics of the surfaces themselves

D. Increases fluid film strength resulting in greatly reduced wear while imparting extreme pressure properties (EP)







Roller bearing uses Steel Shield

IO. ADVANTAGES & TARGETED INDUSTRIES

I. Concept

Van der Waals Forces **Dipole-Dipole Surface Reactions**

2. Advantages

Reduces Friction and Wear Provides Smoother Operation Improves Lubrication Non-Toxic and Helps Build Green Environment Improves Machinery Functionality Improves Fuel Economy Reduces Operating Temperatures Protects Moving Metal Parts Eliminates Cold Start Problems Reduces Maintenance & Downtime Extends Component Reliability & Parts Life

4. Targeted Industries

Automotive & Racing, Airlines & Ground Equipment, Rail & Mass Transit, Shipping Gas, Oil & Energy Industries, Mining & Drilling Lifts, Air Conditioning & Cold Storage Systems Industrial, Agriculture, Construction & Naval Engineering Military & Law Enforcement units







II. HVAC SYSTEMS & LUBRICATIONS

"The need to cool specific environments to either keep food from becoming rancid or to keep us comfortable during the summer requires the use of a HVAC system. The warm air passes through the indoor evaporator coil and the heat energy from the air transfers to the refrigerant inside the coil. The refrigerant absorbs the heat from the air. As a result, the air becomes "cool" and is circulated back providing comfort and cold storage. The refrigerant is pumped back to the COMPRESSOR where the heat absorbed by the refrigerant is released and cycle begins again."

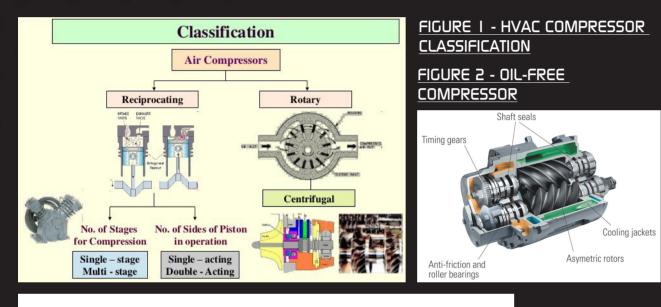
"LUBRICATION is an important aspect of this process, as the components in a compressor need to be maintained in a hermetic environment. The lubricant also must be compatible with a volatile liquid or refrigerant used by the compressor to remove heat from the system."

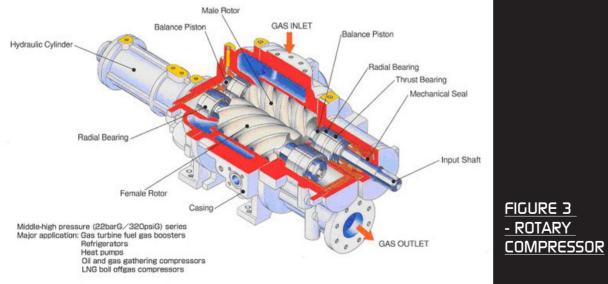


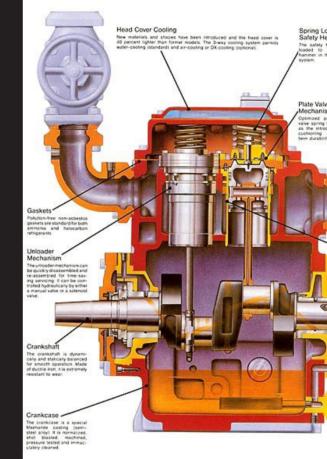
I IA. COMPRESSORS AND LUBRICATIONS

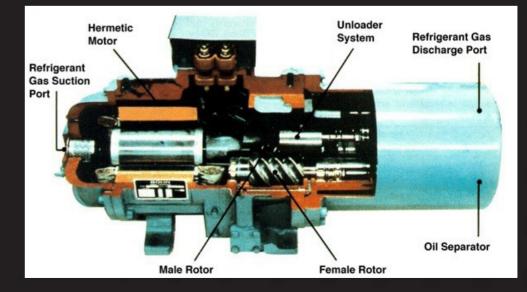
fluid, typically air, or a host of other gases. Dynamic compressors are based on the principle of imparting velocity to a gas stream and then converting this velocity energy into pressure energy. In contrast, positive displacement compressors confine a certain inlet volume of gas in a given space and subsequently elevate this trapped amount of gas to some higher pressure level. Majority

A compressor elevates the pressure of a compressible process of the compressors in either the dynamic (axial/centrifugal) or positive displacement (reciprocating and screw-type) category incorporate moving components. All compressors require lubrication except static jet compressors (ejectors) and the late 20th century oil-free machines with rotors suspended in magnetic or air bearings.









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1 mar	Cylinder Sleeves
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1	Oil Filtration
	The simply-designed Cuno filter (standard) provides dependable foliation. An external of filter is

of cooler is

FIGURE 4 - RECIPROCATE COMPRESSOR

FIGURE 5 - SCREW COMPRESSOR



Let's deal with the lubrication of dynamic compressors (Figure 6). Dynamic compressors require a coolant/lubricant for the gears, bearings and seals. Majority of the dynamic compressors utilize oil film-lubricated seals, as illustrated in Figures 7d, 8a and 8b. Only labyrinth seals (Figures 7a and 7b) or gas-lubricated seals (Figure 8c) operate without a liquid film separating the faces. The bearing and sealing lubricant are often the same on the more conventional liquid- lubricated seals.

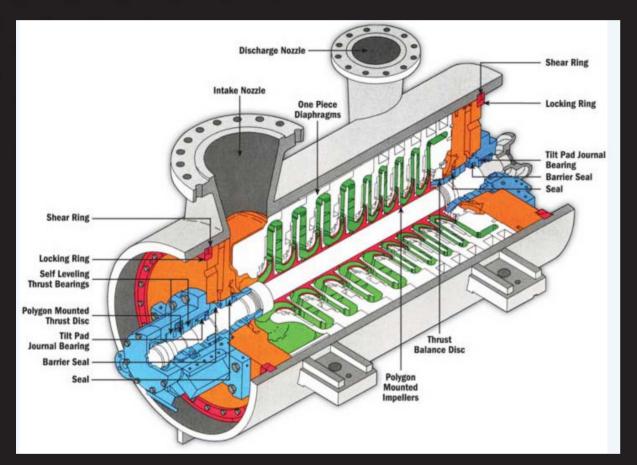


FIGURE 6 (UPPER) - MULTISTAGE BARRELL-TYPE CENTRIFUGAL COMPRESSOR

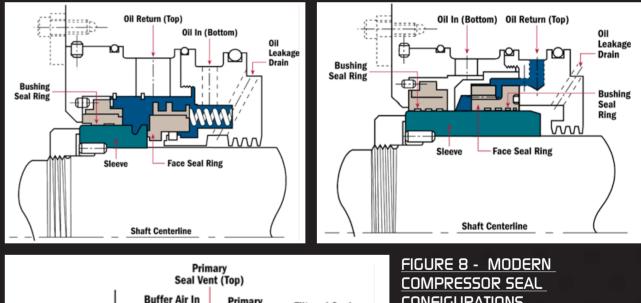




FIGURE 7 - TRADITIONAL COMPRESSOR SEAL DESIGNS • 7a (Left) & 7b (Right): labyrinth seals







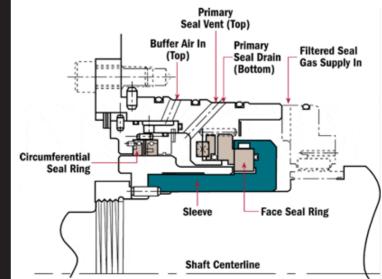




FIGURE 7 (CONTINUED) -TRADITIONAL COMPRESSOR SEAL DESIGNS

- 7c (Top Left)
 7d (Top Right): oil film-lubricated seals

CONFIGURATIONS

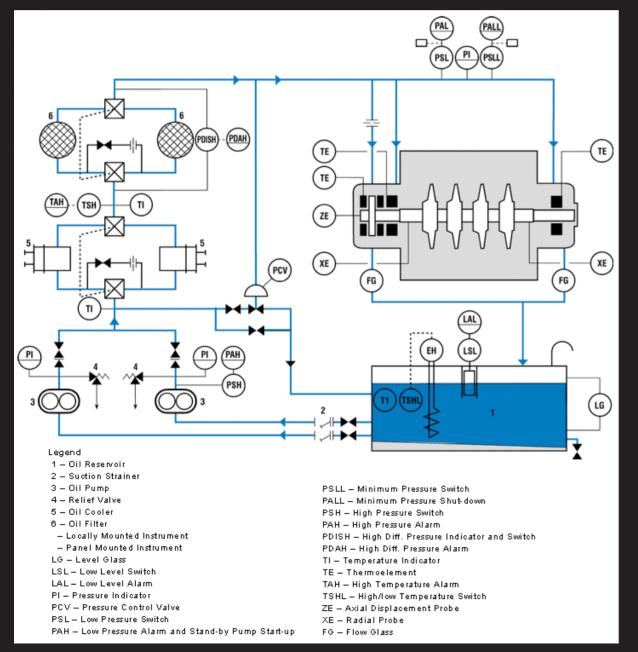
- 8a (Middle Left)
 8b (Middle Right)
 8c (Bottom Left)



LUBRICATING OIL SYSTEM OPERATION

The lube oil system (Figure 9) supplies oil to the compressor and driver bearings and to the gears and couplings. The lube oil is drawn from the reservoir by the pumps and is fed under pressure through coolers and filters to the bearings. Upon leaving the bearings, the oil drains back to the reservoir.

FIGURE 9 - LUBRICATING OIL SYSTEM **OPERATION**



The reservoir is to permit circulation of its entire fluid volume between eight to 12 times per hour. The reservoir may be pressurized or vented.

When in operation, the compressor lubricant oil is normally circulated by the main oil pump. An auxiliary pump serves as a standby. These two pumps generally have different types of drive or power sources. On compressors with step-up gearboxes, the main oil pump may be driven mechanically from the gearbox, and the auxiliary pump operates during the start-up and run-down phases of the compressor train.

Heat generated by friction in the bearings is transferred to the cooling medium in the oil coolers. Air-cooled oil coolers may be employed as an alternative to water-cooled oil coolers. A pressureregulating valve is controlled by the pressure downstream of the filters and maintains constant oil pressure by regulating the quantity of bypassed oil.

A pressure switch activates the auxiliary oil pump. If the oil pressure falls below a preset limit, a second pressure switch shuts down the compressor train. Filters clean the lube oil before it reaches the lubrication points and a differential pressure gauge monitors the degree of fouling (flow restriction) of the filters.

The flow of oil to each bearing is regulated individually by orifices, particularly important for lubrication points requiring different pressures. Lube oil for the driver and other mechanical components is taken from branch lines.

Temperatures and pressures are measured at all important locations in the system, including temperatures from oil sumps, return lines from bearings, gears and other mechanical components. Temperatures and pressures are often recorded on the suction and discharge sides of each compression stage to offer the operator a sense of the health of the system. The readings can be taken locally or transmitted to a monitoring station.

COMPRESSOR SEALS

In general, the mechanical contact or oil face seal (Figure 8a) employs a spring-loaded stationary carbon ring in sliding contact with a rotating ring. This type of seal is also effective when the compressor is at standstill and the oil pumps have been shut down.

The main components of oil bushing seals (Figure 8b) are two stationary, but radially free-to-move (floating ring) breakdown bushings with small diametral clearances opposite a shaft sleeve (Figure 8b). The floating ring clearance controls the flow of the seal liquid cooling the seal. Floating carbon ring seals (not shown) successfully combine some of the best features of all of the above. They, too, require seal face lubrication.



SEAL OIL SYSTEM OPERATIONS

The seal oil system (Figure 10) supplies the mechanical contact and floating ring seals with an adequate flow of seal liquid ensuring proper function. An effective seal is provided at the settle-out pressure when the compressor is not running. The seal oil system may be combined with the lube oil system if the gas does not adversely affect the lubricating qualities of the oil, or provided the oil made unserviceable by the gas does not return into the oil system.

There are two methods of combining lube oil and seal oil systems: booster or combined systems. The hardware and operation of each of these types of oil systems are identical or nearly identical. Mechanical face seals and floating ring seals are supplied with seal oil at a defined differential pressure above the reference gas pressure (pressure within the inner seal drain). The flow of seal oil is regulated by a differential pressure-regulating valve, which changes the pressure of the seal oil relative to changes in system gas pressure or, as shown in Figure 10, by a level-control valve that maintains a constant level in the overhead tank.

The oil in the overhead tank is in contact with the reference gas pressure via a separate line, with a static head providing the required pressure differential. In addition, the oil in the overhead tank compensates for pressure fluctuations and serves as a rundown supply if pressure is lost. If the level in the tank falls excessively, a level switch shuts down the compressor. A moderate oil temperature is maintained by a constant flow of oil through the overhead tank.

COMPRESSOR LUBRICANTS

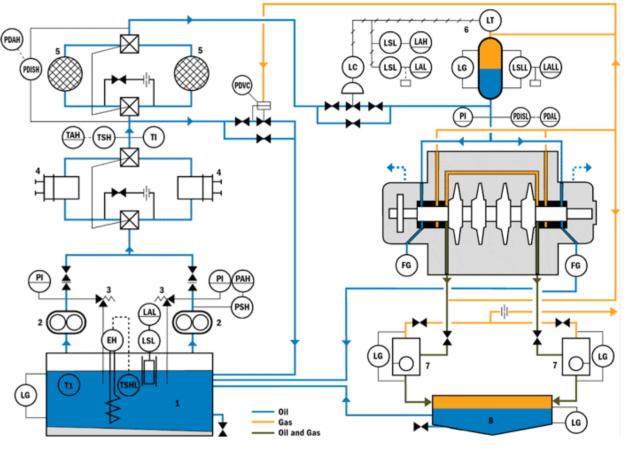
Majority of the compressors are best served by premium-grade turbine oils with ISO viscosity grades of 32 or 46. Premium-grade ISO VG 32 turbine oils are used more often than the heavier viscosity grades. Long-term lifecycles are associated with premium-grade product selection, large sumps, reasonably good contamination control and the occasional top-off "sweetening" effect on the oil in use.

For the mechanical contact seal system, a regulating valve maintains the reference gas and the seal oil at a constant differential pressure. As the name indicates, the mechanical contact seal serves as a mechanical standstill seal when the compressor plant is shut down.

The seal oil is split into two streams in the compressor seals. Most of the flow returns under gravity to the reservoir. A small quantity passes through the inner seal ring to the inner drain, where it is exposed to the gas pressure.

This oil, mixed with the buffer gas, flows to the separator system, which consists of a separator and a condensate trap on each side. The separated gas flows to either the flare stack or to the suction side of the compressor while the oil flows into a tank for further decassing.

If oil is used as sealing liquid and can be used again, degassing is accelerated by heating or by air or nitrogen sparging. Sparging units perform on-stream purification of oil which can keep lubricants serviceable for long time periods. Only if the oil becomes unusable is it led away for separate treatment or disposal. The quantity of oil passing through the inner drain in a modern centrifugal compressors is small and ranges from 5 to 50 liters per day on new machines.



Legend 1 - (

1 - UH Reservoir	LAH
2 – Oil Pump	LSLL
3 – Relief Valve	LALL
4 - Oil Cooler	PALL
5 – Oil Filter	PI –
6 – Overhead Seal Oil Tank	PSH
7 – Seal Oil Trap	PAH
8 – Drain Pot	PDIS
 Locally Mounted Instrument 	PDA
 Panel Mounted Instrument 	PDIS
LG – Level Glass	PDA
LSL – Low Level Switch	TI — 1
LAL – Low Level Alarm	TSH
LT – Level Transmitter	TAH
LT – Level Transmitter	TSHL
LC – Level Controller	FG –
LSH – High Level Switch	

- Flow Glass

32

LAH – High Level Alarm

L – Minimum Level Switch

L – Minimum Level Shut-down

L – Minimum Pressure Shut-down

Pressure Indicator

- High Pressure Switch – High Pressure Alarm

ISL – Diff. Pressure Indicator and Switch

AL - Low Diff. Pressure Alarm.

SH – High Diff. Pressure Indicator and Switch

AH – High Diff. Pressure Alarm

Temperature Indicator

– High Temperature Switch

– High Temperature Alarm

IL – High/low Temperature Switch

FIGURE IO - LUBRICATING OIL SYSTEM OPERATION



I 2. TECHNICAL FACTORS AFFECTING LUBRICATIONS IN HVAC SYSTEMS

"STEEL SHIELD ABF Technology is specially formulated to protect and prolong the life of the compressor unit and to restore the lost efficiency of the heat exchanger system due to fouling. This results in less maintenance and down time for HVAC&R equipment and reduces operating costs. STEEL SHIELD lubricants are completely inert relative to all refrigerants and their respective lubricants and will not affect filter performance in any way. Now, use STEEL SHIELD on your commercial and residential air conditioners, refrigeration systems, heat pumps, ammonia and non-ammonia systems."



I 2A. REFRIGERATION AND AIR CONDITIONING INTRODUCTION

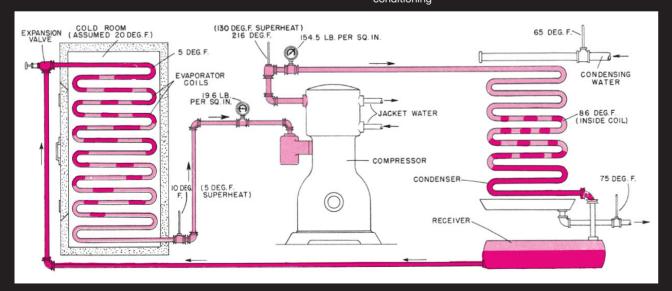
The basic principles of the refrigeration compression cycle are shown in Figure 1. The five essential parts basic to every system are shown: evaporator, compressor, condenser, receiver, and expansion valve (or capillary). Liquid refrigerant flows from the receiver under pressure through the expansion valve to the evaporator coils, where it evaporates, absorbing heat and resulting in a cooling action. The vapor is then drawn into the compressor, where its pressure and temperature are raised. At the higher pressure in the discharge of the compressor, the condensing temperature of the refrigerant is higher than it would be at atmospheric pressure. When the hot, high pressure vapor flows from the compressor to the condenser, the cooling water (air in some applications) removes enough heat from it to condense it. The heat removed from the refrigerant in the condenser is equal to the amount of heat removed from the cold room (cooling action) plus the heat resulting from the mechanical work done on the refrigerant in the compressor that is not removed by the jacket cooling of the compressor. In many commercial installations, the evaporator cools a heat transfer fluid such as brine, which is then pumped through the area to be cooled. Smaller units, such as home refrigerators and freezers, room air conditioners, and automotive air conditioners, have air-cooled rather than water-cooled condensers.

In commercial installations, two or three stages of compression may also be used. If system pressures or cooling capacities dictate the use of two stages of compression, two-stage compressors are used, or a combination of separate single-stage compressors. Rotary sliding vane, scroll, or rotary screw compressors are sometimes used at low to moderate pressures or for booster purposes. Multistage reciprocating compressors are used for large air conditioning installations, with a trend toward the use of more scroll compressors. Reciprocating compressors are commonly used for refrigeration systems, with a trend toward the use of rotary vane. Centrifugal compressors are also used on some commercial refrigeration systems as well as in chillers. Reciprocating, sliding vane, and scroll compressors are used for automotive air conditioning systems, with some screw and axial piston compressors also used. Some very small units such as dehumidifiers may be equipped with diaphragm-type compressors. Reciprocating compressors are used in most other applications.

Most reciprocating compressors for commercial installations are of the single-acting, trunk piston type and have closed crankcases. As a result of refrigerant leakage past the pistons, the crankcases are filled with a refrigerant atmosphere. The same is true of axial piston units used for automobile air conditioning. Crosshead and double-acting compressors have open crankcases. The majority of small to medium-sized electric motor driven refrigeration and air conditioning units are hermetically sealed, with all the operating parts, including the electric motor, inside the sealed unit. Evaporators may operate either dry or flooded. In dry evaporators, only refrigerant vapor is present, while flooded evaporators have both liquid and vapor present.

FIGURE I - BASIC SINGLE-STAGE COMPRESSION REFRIGERATION SYSTEM

 The elements shown are common to all compression refrigeration systems, whether refrigeration or air conditioning

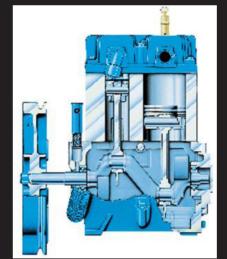


I 2B. COMPRESSOR FACTORS AFFECTING LUBRICATION

CYLINDER CONDITIONS

The oil film on the cylinder walls of a reciprocating refrigeration compressor is subjected to low temperatures at the suction ports and to moderately high temperatures near the cylinder head. Since viscosity decreases with temperature, the oil near the suction ports will have considerably higher viscosity than the oil near the cylinder head. Nevertheless, the oil must spread in a thin film over the entire working surface. Spreading is accomplished by the piston rings (or the piston itself in small compressors without piston rings) as the pistons move back and forth. The oil must distribute rapidly, but to do this it must not be too high in viscosity. On the other hand, an oil too low in viscosity will not protect against wear.

Oil carried out of the cylinders to the valves and discharge piping is subjected to the temperature of the discharging refrigerant. Ordinarily the temperature of the discharging refrigerant is not high; for example, the discharge temperature of a single-stage ammonia









compressor with a compression ratio of about 51 should not be much in excess of 250F (121C). Some single-stage units operate at higher ratios, and higher discharge temperatures; but in most small compressors, the valve temperatures remain moderate because of the relatively large cooling area in proportion to cylinder volume. The discharge temperatures of compressors operating on fluorocarbon refrigerants are lower than those of equivalent machines operating on ammonia, although the compressors of automobile air conditioning systems may operate at quite high discharge temperatures.

When two or more stages of compression are used, the operating temperature in each stage usually is lower than in single-stage machines. In rotary compressors, the discharge temperatures are also usually moderate because of low compression ratios.



FIGURE I - COMMON

COMPRESSOR TYPES

- Top Left: Reciprocate compressor
- Top Right: Centrifugal compressor
 Bottom Left: Screw rotary compressor
- Bottom Middle & Right: Scroll compressor
- Bottom Middle & Right: Scroll compressol







12C. HVAC SYSTEM FACTORS AFFECTING LUBRICATION

OXIDATION

In compressors with enclosed crankcases, temperatures are normally moderate and the entire machine is filled with refrigerant vapor. Very little, if any, air is present. Under these conditions, oxidation in the usual sense does not occur, although it is doubtful that it can be avoided entirely. Limited oxidation does not impair the lubricating value of an oil because the initial oxidation products formed are soluble in the oil. If oxidation progresses too far, eventually some of the soluble oxidation products become insoluble when the oil is cooled. These products could plug or restrict capillary tubing or orifices inside the system.



FIGURE 2 - OXIDATION OF A COMPRESSOR INTERNAL SURFACE

BEARING SYSTEM CONDITIONS

The general requirements of the bearing systems of refrigeration compressors are similar to those of other comparable compressors. However, some special factors must be considered. In the compression of air or gases such as hydrocarbon gases, it is desirable that the oil not be miscible with the gases, whereas in closed refrigeration systems, the oil must be somewhat miscible with the oil to be able to circulate throughout the system and get to all the components in need of lubrication.

In compressors with closed crankcases, there is very little exposure to oxygen, and thus oxidation stability of the oil is not a major concern. If the same oil is used for both bearings and cylinders, as in many small units, however, the oil must have oxidation stability adequate for the cylinder conditions.

When ammonia is used as the refrigerant in compressors with closed crankcases, any additives used in the oil must be types that are not affected by ammonia. A refrigerant that is soluble or partially soluble in the oil, as are the majority of the fluorocarbon refrigerants, will dilute the oil and reduce its viscosity, a sequence of events that must be considered in the selection of the oil viscosity.

The motor in a hermetically sealed unit is completely surrounded by a mixture of refrigerant and oil. Thus the oil must have good dielectric properties, must not affect the motor insulation, and must not react with the copper motor windings or other system materials at elevated temperatures. Since most such units are operated on fluorocarbon refrigerants, the dilution effect of the refrigerant on the viscosity must be considered.

When the crankcase and cylinders are completely isolated from each other, as in compressors having crosshead construction, the oil in the crankcase is exposed to air and there is intimate mixing of the warm oil with air. These conditions are favorable to oxidation and require a chemically stable oil to resist oxidation.

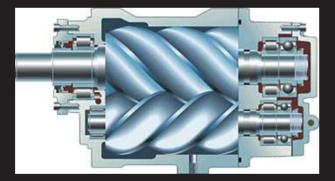




FIGURE 3 -COMPRESSOR BEARINGS Top: General bearing locations of a screw compressor Bottom: Roller bearings application in a screw components

If the oil carried out of the compressor cylinder forms gummy deposits in the condenser, or congeals or forms waxlike deposits in the evaporator, capillary tube, or expansion valve, there may be serious reduction in the heat exchange capacity. Heat insulating deposits in the evaporator make it necessary to carry a lower evaporator temperature to produce the required refrigeration effect. This in turn requires a lower evaporator pressure and increases the power required by the compressor for a given refrigeration duty, owing to the increased pressure range through which the gas must be compressed. In addition, at the lower suction pressure, the vapor density is lower, forcing the compressor to handle a greater volume With the refrigerants that are miscible or partly miscible with of vapor and thus reducing refrigeration capacity. Heat insulating deposits in the condenser increase the temperature difference between the cooling medium (water or air) and the condensing refrigerant. The resulting higher condensing temperature makes higher compression necessary and increases power consumption.

Whether heat insulating deposits will be formed depends on the properties of the lubricating oil, the refrigerant in use, the evaporator temperature, and the equipment used in the system. The effects of some of the common refrigerants are considered separately.

FLUOROROCARBONS

Chlorofluorocarbons (CFCs) are being phased out for use in The waxy materials that precipitate from these oil-refrigerant air conditioning and refrigeration systems because of their mixtures can also clog expansion valves and capillary control potential negative effects on the ozone layer. As a result, more tubes, preventing their proper functioning. However, the environmentally friendly non-CFC refrigerants are being developed. concentration of oil in the refrigerant at the expansion value is Several alternative refrigerants have been around for many years, usually lower than in the evaporator, so the floc point is depressed such as ammonia, hydrocarbons, carbon dioxide, methyl chloride, below what it would be at a 10% concentration. As a result, if and others that do not pose problems from the ozone depletion the oil selected has a low enough floc point for conditions in the standpoint. These will be continued to be used in many applications. evaporator, it usually will not cause difficulties in the expansion Non-CFC fluorocarbon refrigerants such as R-134a, R-123, and valve or capillaries. Difficulties in these areas attributed to mineral blends such as R-404A. R-407C. and R-410A are replacements oils are frequently due to ice crystals formed by minute quantities for the CFCs. The use of these alternative refrigerants is increasing of water in the system. rapidly.

Each of the alternative materials has specific properties and operating characteristics that must be understood and handled appropriately to ensure maximum system performance as well as the safety of the people working with them and the public potentially exposed to them. In many systems, CFCs are and will remain in service. The Montreal Protocol banned the production of CFCs as of January 1, 1996, and hydrochlorofluorocarbons (HCFCs) were limited to production levels as of the same date, with a cease production date in the year 2030.

Air conditioners in older automobiles, as well as, many home refrigerators and air conditioners, were filled with CFCs, and many of these units are still in service. When systems containing CFCs need servicing, they must be refilled with CFCs manufacture before January 1, 1996, use reclaimed CFCs from older systems, or retrofit the systems to accept R-134a or one of the alternative environmentally friendly refrigerants. Gradually, all the CFCs and HCFCs will be replaced by alternative HFC materials, as well as by other gases such as isobutane, propane, and ammonia.

oil, enough of the refrigerant dissolves in the oil to depress the pour point of the oil sufficiently to prevent congealing of the oil on evaporator surfaces in most cases. However, there is a temperature at which a heavy, flocculent precipitate first appears when a mixture of Freon 12 and 10% of the oil is chilled. The temperature at which this occurs depends on the refrigerant, the percentage of oil in the refrigerant, and on the oil. Refrigeration systems using fluorocarbon refrigerants are often designed to ensure that approximately 10% oil is present in the evaporator. In some cases, the evaporator is actually charged with this amount of oil. Under these conditions, the floc point of the oil (also known as the critical separation temperature) represents the lowest temperature that can be used with that oil.

Oil selection can go a long way to minimizing problems related to lubrication in systems using fluorocarbons. The use of highly refined naphthenic or paraffinic mineral oils works satisfactorily with both the CFCc and HCFCs. The base stocks for these oils are usually severely hydroprocessed or acid-treated to remove wax and other materials undesirable from a refrigeration oil standpoint. For HCFCs, alkyl benzene synthetic lubricants provide excellent performance, as does mineral oil. Widely used products for HFCs are polyol esters, polyalkylene glycol, and polyvinyl ether.



AMMONIA. CARBON DIOXIDE

Oil is slightly miscible in anhydrous ammonia and carbon dioxide. Generally, not enough of the gas dissolves in the oil to have a significant effect on the pour point of the oil. Thus, if the pour point of the oil is above the evaporator temperature, oil will congeal on the evaporator surfaces and form an insulating film that interferes with heat flow and efficient performance of the system. To remove the oil, the evaporator must be periodically warmed, liquefying the oil so that it will drain from the surfaces to a location from which it can be removed. With flooded evaporators, refrigerant flow may be so rapid that there is little or no opportunity for the oil to collect on evaporator surfaces, and the pour point of the oil may not be a major concern. Ammonia is not compatible with copper or brass and cannot be used in systems containing these metals. As with CFCs and HCFCs, ammonia works well with highly refined mineral oil. CFCs and HCFCs also can use polyalphaolefins (synthesized hydrocarbons), polyalkylene glycols, and polyol esters.

HYDROCARBON REFRIGERANTS

Isobutane and propane gases are being used as replacements for CFC refrigerants in some applications. These gases are primarily used in smaller units such as hermetic household refrigerators.



FIGURE I - ANSI/ASHRAE STANDARD 34-

2010

Designation and Safety Classification of Refrigerants

SULFUR DIOXIDE

Sulfur dioxide has a selective solvent action that with conventional lubricating oils results in sludge. It, therefore, requires the use of highly refined white oils or group III base stocks with low levels of additive.



LUBRICATING OIL RECOMMENDATIONS

Table 1 shows general lubricant recommendations by refrigera type. The lubricants are classified according to base type. requirements of oils for refrigeration systems can be summariz as follows.

- 1. The oil should be of proper viscosity to distribute readily at the system's lowest temperatures yet provide adequate films to protect against wear in the cylinders and crankcases.
- 2. The oil should have chemical stability adequate to resist oxidation and the formation of deposits in crankcases open to the atmosphere, and to resist the deteriorating influence of high temperatures at compressor discharge.
- 3. In closed systems without oil separators, the oil should be miscible with the refrigerant, to ensure that the oil will circulate through the system and return to lubricate the compressor. In closed systems with separators, it is

Refrigerant	Mineral Oil ^A	Synthetic ^B
Fluorocarbons		
CFC- 11, 12, 113, 114, 500, 502	Yes	PAO, POE
HCFC- 22, 123, 125, 408A (blend)	Yes	PAO, AB
HFC- 134a, 143a	No	POE, PAG, PVE
Blends 404A, 407C, 410A	Yes	POE, PVE
Ammonia	Yes	PAO, PAG, POE
Carbon dioxide	Yes	PAO



ant	desirable that the oil not be miscible with the refrigerant,
'ne	to facilitate separation. In open crankcase systems, it is
ed	desirable that the oil not be soluble or miscible with the
	refrigerant, to minimize dilution.
	4. The oil should be able to withstand system temperatures
	without breakdown, and it should not inhibit the heat transfe

- characteristics of the refrigerant. 5. The oil must be chemically stable and must not react with the refrigerant or system components. Some additives in the
- oil can react with the refrigerant to form deposits or sludges.
- 6. The oil must reduce friction and minimize wear.
- 7. The oil must keep the system clean and stay in service for extended intervals.

Oil viscosities recommended vary from as low as ISO VG 7 to as high as ISO VG 150.

Lubricating Oil

TABLE I - LUBRICATING OIL **RECOMMENDATIONS BASED** ON REFRIGERANTS

Notes:

- A: Mineral oils are to be highly refined paraffinic or naphthenic. White oils or severely hydroprocessed base stocks should be used.
- B: PAO, Polyalphaolefin; POE, polyolester; AB, alkylbenzene; PAG, polyalkylene glycol; PVE, polyvinyl ether.



I 3. STEEL SHIELD APPLICATIONS IN HVAC

STEEL SHIELD lubricants have a wide range of applications in the HVAC industry. Its unique ABF Technology will improve the function and reliability of all metal components of the entire system without changing its integrity or causing any undesirable effects. SwRI test reports proved that STEEL SHIELD lubricants are superior to the premium brands such as Mobil, Shell and Caltex etc.

STEEL SHIELD refrigeration lubricant lubricates internal parts, removes heat generated during compression, cleans the system, acts as a fluid seal and **reduces energy** requirements.

STEEL SHIELD boosts antiwear protection, metal passivation and acid scavenging.

STEEL SHIELD enhances **heat-transfer and compressor efficiency**. Other functions are to control foam and **reduce noise** generated by moving parts of the compressor.'

"STEEL SHIELD ABF Technology is the only solution for all HVAC Systems."



APPLICATIONS OF STEEL SHIELD

The ABF Technology of Steel Shield lubricants treats the metal surface. It flattens and rolls out metal surface and greatly reduces friction and metal wear of any mechanical system. Therefore, the pistons, screws, rotary parts or any types of bearing can last much longer, and energy lost due to frictions decreases dramatically.

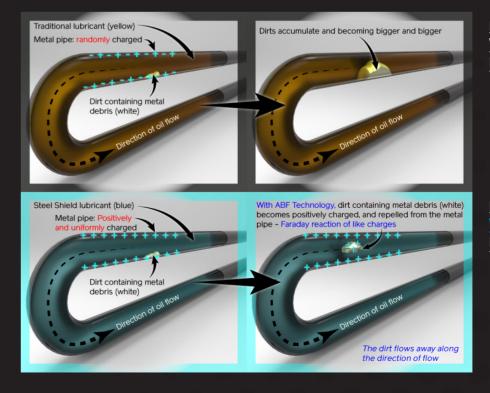
The work load of electric motors and any power system reduced, and hence increase in efficiency.

When Steel Shield lubricants are applied continously, the system can achieve:

- 1. Reduces energy consumption by 3 to 18%
- 2. Protects metals during operation, and suppresses maintenance cost to minimum
- 3. Enhances and extend the life of metal components like bearings to 2 times or much more
- 4. Dramatically reduces noise and keeps system slience during operations
- 5. Reduces heat generation of the system and keeps energy consumptions to minimum
- 6. Boosts the entire HVAC system efficiency to maximum

NON CORROSIVE CLEANSING IN HVAC

Steel Shield lubricants utilizes ABF Technology can positively charges BOTH the inner surface of metal pipes / tubes and the metal-containing sludge. The results are cleansing / repelling of



existing sludge on the tube surface and the prevention of the new sludge formations. These greatly improves the overall efficiency of HVAC system by enhancing the heat conduction efficiency.

FIGURE I - HVAC PIPE WITHOUT TRADITIONAL LUBRICANT

FIGURE 2 - HVAC PIPE WITH STEEL SHIELD LUBRICANT



I 4. DECRYPTION OF CENTRAL HVAC SYSTEM CONSUMPTIONS

"The goal of using STEEI SHIELD lubricant is to reduce the amount of energy required to provide products and services. With increasing demands and company's economy of scale, it is possible to cut a huge amount of maintenance cost with STEEL SHIELD products."



14A. LET STEEL SHIELD TELLS YOU THE TRUTH!

WHY THE CENTRAL AIR-CONDITIONING SYSTEM COST IS HUGE?

A Perspective on the Power Consumption of a Central Air-Conditioning Plant. Very often most of the building owners only know from the accounting figures that they are paying hundreds and thousands of dollars for the electricity fees each month. They are unware of the fact that a large portion of the electricity fees is for the power consumption of the central air-conditioning plant of the building. In order to have a clearer picture on the amount of money spent on air-conditioning electricity fees, you are invited to spend a few minutes and go through the following context.

It would be natural for building owners to focus on the leasing part of business ever since the opening of the new building. However, after the building is occupied, a large amount of money is spent on the various building facilities, such as management fees, electricity fees, water fees, etc. It is always the case that electricity fees is the highest of all other fees. And out of this large portion of electricity fees, believe it or not, 70% or more belongs to the central air-conditioning plant while the other 30% is for lighting, lifts, controls, etc.

Now let us take a further step and try to work out an estimation on the expenditure of the power consumption of a central A/C plant. Several assumptions are made for calculation purpose:

- 1. Cooling capacity of the plant is 500TR
- 2. Operating 12 hours a day
- 3. Chiller efficiency is 1.5Kw/TR and
- 4. Overall average operating load is 75% accounting for chillers down time and times when chillers run below full load.

At this moment, it is perhaps worthy of pointing out that the compressor motor of most of the chillers are of the constant speed type. In fact they are drawing 70% or more of the full load current (FLA) even though running at as low as 50% of it's cooling capacity. Which means it would be fair enough to assume that the chillers are running at 75% cooling capacity all the year round. This can also be illustrated by the following:

Since, Power (Kw) = Current² (I) x Resistance(R)

i.e. $Kw = I^2 R$ and $I = \sqrt{(Kw/R)}$

Also, Tons of Refrigeration(TR) = $Kw \bullet hr$,

We have $I_2 = I_1 \times \sqrt{(TR_2/TR_1)}$

Therefore, current is proportion to the square root of cooling capacity which is shown in the table using a 300TR chiller:

Capacity	100%	90%	80%	70%	60%	50%
TR	300	270	240	210	180	150
Amp, A	400	379	358	335	310	283
FLA	100%	95%	89%	84%	77%	71%

The amount of electricity fees

- = 500 (TR) x 1.5 (Kw / TR) x 0.94 (\$ / Kw hr)
- x 12 (hr / day) x 30 (day / month) x 12 (month / year)
- x 0.75 (% of capacity)

This is in reality a huge amount of money bearing in mind that this is only the electricity fees for a 500TR A/C plant. We can just imagine what it would be for a 1,000TR or 2,000TR airconditioning plant.....A lot more than we can expect!

= HK\$ 190,350 per month, or (HK\$ 2,284,200 per year)





I 5. STEEL SHIELD ORIGINAL USA ADDITIVES FOR HVAC

STEEL SHIELD EPA





REEL SHIELD GREASE









TOOL SHIELD

STRIKE SHIELD



LITHI SHIELD



SPRAY SHIELD







STEEL SHIELD EPA

STEEL SHIELD Extreme Pressure Anti-Wear (EPA)™ is the ultimate protection for the moving metal parts for industry. Utilizing the most Advanced Boundary Film (ABF) Technology, it protects moving metal parts from heat, friction and wear in engines, transmissions, differentials, transfer cases, hydraulic pumps and motors, gear boxes, and other enclosed lubrication systems, due to boundary conditions of frictional abrasion, extreme pressure torque, dry startup and shutdown. Increased performance and greatly reduced maintenance and downtime are the results. These performance goals are achieved through ABF Technology by lowering the operating temperatures, extending the life of component parts and increasing reliability.



Standard

Properties

Flash Point

Boiling point

Evaporation rate

Vapor pressure

Specific gravity

Unit

°C

°C

Result

226

238

1.07

< 0.01

<1@25 °C

ORIGINAL USA ADDITIVE

Advantages

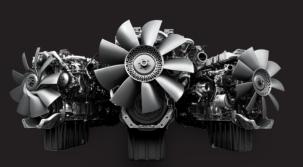
- Dramatically reduces wear
- Extends parts life and component realibity
- Improves lubrication • Protects moving metal parts
- Reduces friction
- Reduces maintenance and downtime • Reduces operating temperatures
- Smoother operation

Applications

- All Engines, Transmissions & Differentials
- Hydraulic Systems
- Open Gears
- Gear Boxes
- Gear Reducers Gear Couplings
- Electric Motors
- Heavy Machinery
- Weapon Systems

Directions of Usage

- Sasoline And Diesel Engines: Add 2 oz. per quart of oil.
 Auto Transmissions: Add 1 oz. per quart of fluid.
- Manual Transmissions & Differentials: Add 2 oz. per guart of
- gear lube/oil.
- Gear Boxes: Add 2-3 oz. per quart. Hydraulics: Add 1 oz. per quart of fluid.
- Contains no volatiles or solvents. Contains synthetic hydrocarbons and advanced chemical additive technology. Non-toxic and environmentally friendly.





ORIGINAL USA ADDITIVE

Advantages

- Cleans & Removes Internal Dirt
- Dramatically Reduces Metal-To-Metal Wear
- Extends Tool Life
- Improves Tool Power & Performance
- Lubricates, Cleans & Protects
- Protects Moving Metal Parts
- Repels Air Line Moisture
- Smooths Tool Operation
- Stops & Inhibits Rust

Applications

- Air Cutting Tools, Air Drills, Air Grinders, Air Nailers, Air Ratchets, Air Sanders, Air Staplers
- Automatic Oilers
- Hand Tools
- Impact Wrenches
- Piston & Rotary Type Air Tools

Directions of Usage

- Use in accordance with tool manufacturer's instructions. • Tools may need to be lubricated daily, or several times a day, depending on the frequency and prolonged use of the
- tool. • Contains no volatiles or solvents. Contains synthetic hydrocarbons and advanced chemical additive technology. Non-toxic and environmentally friendly.

TOOL SHIELD

TOOL SHIELD[™] is the ultimate protection for the moving metal parts for automotive and industrial tools. Utilizing the most Advanced Boundary Film (ABF) Technology, it protects moving metal parts from heat, friction and wear due to boundary conditions of frictional abrasion, extreme pressure torque, air line moisture and internal dirt. It works in all piston and rotarytype air tools, stationary and handheld power tools and many hand tools. Increased power and performance and greatly reduced wear while removing dirt from tool are the results. TOOL SHIELD[™] contains ABF (Advanced Boundary Film) for increased lubricity and boundary film lubrication.



Properties	Standard	Unit	Result
Flash Point (PMCC)		°C	61
Boiling point		°C	186 - 201
Evaporation rate		•	<0.01
Vapor pressure	•	0 (<1@25 °C
Specific gravity	0		1.02





LITHI SHIELD

LITHI-SHIELD[™] is the ultimate in extreme pressure anti-wear lithium complex grease. It exceeds all other lithium complex greases due to the addition of ABF (Advanced Boundary Film)Technology, extreme pressure and antifriction additives added to its formula. LITHI-SHIELD™ treats, seals and smooths metal surfaces to dramatically reduce friction, as well as friction related heat and wear. LITHI-SHIELD's™ unique formulation allows it to exceed the performance of other greases while using smaller quantities. In fact, LITHI-SHIELD[™] exhibits great oxidation resistance, over twice that of its nearest competitor.



ORIGINAL USA NLGI No. 2 GREASE

- Advantages

 Maximum Protection Against Wear And Extreme Pressure • Adheres To Metal Exhibiting Top Performance In Roll
- Stability • Provides Constant Lubrication To All Areas
- Offers The Maximum In Friction Reduction
- Resists Water Washout

Applications

- All Extreme Pressure Applications
- Axles, Bearings, CV Joints, Universal Joints, Chassis Fittings, Conveyors, Pumps, Rotating Machinery • Boat Trailers And Marine Applications
- Heavy Equipment, Mining Equipment, Railroad Equipment

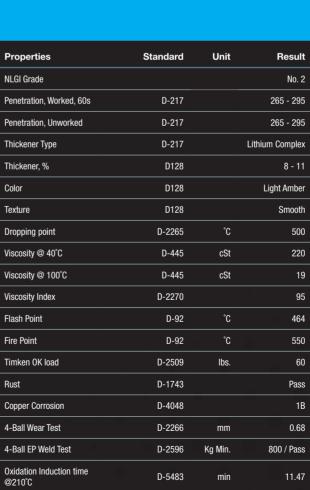
Directions of Usage

• Follow the maintenance manuals of the machines



LITHI SHIELD Outperforms Top Class Greases Made by YAMAMOTO And ATLAS • Check out the "SOUTHWEST RESEARCH INSTITUTE TEST REPORTS" to learn more

Test Item	Four-Ball Extreme Pressure Properties	Lithi Shield	Yamamoto EP Grease	ATLAS CHISEL LUBE
Loading Ability	Corrected Load	851.1	501.68	302.79
Anti-Wear Ability	Load Wear Index	92.27	66.73	41.23
High Temperature Loading	Weld Point	800	315	315
High Pressure Loading	LNSL	80	63	50





Properties	Standard	Unit	Result
Water washout @ 79°C	D-1264		0.027
Mobility at 77°C	US Steel Mobility Test	g/min	576
Mobility at 60°C	US Steel Mobility Test	g/min	275.4
Mobility at 40°C	US Steel Mobility Test	g/min	86.6
Mobility at 20°C	US Steel Mobility Test	g/min	15.3
Mobility at 0°C	US Steel Mobility Test	g/min	1.6



REEL SHIELD GREASE

The ultimate lubricant, cleaner, penetrant, and saltwater protectant, has been aggressively designed and formulated for the Sport Fishing Industry. Reel Shield Grease[™] lubricates and protects against extreme pressure and wear in all moving metalto-metal parts, in all types of fishing reel and drag systems. Reel Shield Grease™ penetrates to the internal moving parts and shields against corrosion in extreme saltwater environments better than any other



the ultimate tool in the total care and maintenance of all fishing tackle in both fresh and saltwater fishing. Reel Shield Grease[™] has been tournament tested in harsh saltwater conditions and proved to be superior in its performance.

Additional testing has proven Reel Shield Grease™ improves casting distances due to its Advanced Boundary Film (ABF) Technology, which reduces coefficients of friction between the gears and other moving metal parts in the reel and roller guides of the rod allowing for smoother casting and overall performance and operation.

ORIGINAL USA NLGI No. I GREASE

- Advantages Dramatically Reduce Gear Water
- Extends Life Of Fishing Pliers, Tools And Equipment
- Extends Reel Life
- Gears Run Smooth & Quiet
- Longer Conventional Casts
- Protects Against Salt Water, Rust & Corrosions
- Smoother Reel Operation

Applications

- All High Temperature Applications
- All Extreme Pressure Applications
- Axles
- Bearings
- Boat Trailers And Marine Applications
- Chassis Fittings
- Conveyors
- CV Joints
- Heavy EquipmentMining Equipment
- Pumps
- Railroad Equipment
- Reel, Ship Equipments
- Rotating Machinery
- Universal Joints

- Directions of Usage
- Use in accordance with machine / tool manufacturer's instructions.
- Apply to bearings and moving parts. Coat metal surfaces lightly and wipe excess off.
- Contains synthetic hydrocarbons. Non-Toxic. Contains no volatiles. If swallowed, do not induce vomiting due to aspiration in lungs.



Properties	Standard	Unit	Result	Properties	Properties Standard	Properties Standard Unit
NLGI Grade			No. 1	Oxidation Induction time	Ovidation Induction time	
Penetration, Worked, 60s	D-217		310 - 340	@210°C	@210°C	@210°C
				Water washout @ 79 °C	Water washout @ 79 °C D-1264	Water washout @ 79 °C D-1264
Penetration, Unworked	D-217		310 - 340	Mobility at 77 °C	Mobility at 77 °C US Steel Mobility Test	Mobility at 77 °C US Steel Mobility Test g/min
Thickener Type	D-217		Lithium Complex	Mobility at 60 °C	Mobility at 60 °C US Steel Mobility Test	Mobility at 60 °C US Steel Mobility Test g/min
Thickener, %	D-128		6 - 8	Mobility at 40 °C	Mobility at 40 °C US Steel Mobility Test	Mobility at 40 °C US Steel Mobility Test g/min
Color	D-128		Light Amber	Mobility at 20 °C	Mobility at 20 °C US Steel Mobility Test	Mobility at 20 °C US Steel Mobility Test g/min
Texture	D-128		Smooth	Mobility at 0 °C	Mobility at 0 °C US Steel Mobility Test	Mobility at 0 °C US Steel Mobility Test g/min
Dropping point	D-2265	°C	500			
Viscosity @ 40 °C	D-445	cSt	220			
Viscosity @ 100 °C	D-445	cSt				
Viscosity Index	D-2270		95			
Flash Point	D-92	°C	464			
Fire Point	D-92	°C	550	Ť		
Timken OK load	D-2509	lbs.	60+			
Rust	D-1743	0	Pass			
Copper Corrosion	D-4048		1B			
4-Ball Wear Test	D-2266	mm	0.7			
4-Ball EP Weld Test	D-2596	Kg Min.	800 / Pass			

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product to date. This distinguishes Reel Shield[™] as





STRIKE SHIELD

STRIKE SHIELD[™] is the ultimate penetrant to rapidly pierce rusted and corroded metal surfaces using a distinctive spreading action to break lose frozen mechanisms while at the same time applying an advanced lubricating film to the surfaces of the metal delivering the highest quality lubrication available in penetrating oil. STRIKE SHIELD[™] leaves a unique layer of film on surfaces that helps prevent rust and corrosion along with driving out and dispersing moisture on ignition wires, electrical contacts, circuit boards and other electrical connections to provide protection against future corrosion in extremely tough conditions.



ORIGINAL USA ADDITIVE

Advantages

- Offers extremely fast penetration and lubrication into remote rusted /corroded areas especially for industrial and marine applications that have seized metal mechanisms
- Provides a quick durable long lasting lubricating film to a variety of different areas including mechanisms in extreme salt water environments
- Inhibits rust and oxidation on metal contacts and surfaces in all weather conditions
- Maximum performance as a moisture displacement on wet electrical switches/boards and electronic systems
- Helps start damp engines by dispersing moisture on ignition wires and electrical systems
- Protects circuit boards from corrosion in all weather conditions including salt spray
- Repels dirt and dust build-up
- Mild and pleasant fragrance

Applications

- Frozen or scaled nuts and bolts
- Sticky locks
- Squeaky hinges Sliding doors
- Wheels
- Conveyors
- Cables
- Linkages, Shafts, Bushings • Sliding parts and mechanisms
- Any automotive, marine, farming industrial or commercial application that requires a fast acting penetrate, lubricant and moisture displacement all combined in one product

Directions of Usage

• Spray to any frozen mechanisms

Properties	Standard	Unit	Result
Flash Point (PMCC)		°C	61
Boiling point		°C	186 - 201
Evaporation rate			<0.01
Vapor pressure			<1@25 °C
Specific gravity			1.02

"STRIKE SHIELD delivers an all in one product that is a fast acting penetrant, extremely durable lubricant and long-lasting rust and corrosion protectant even in tough industrial and harsh salt water environments."





ORIGINAL USA ADDITIVE

- Advantages Creeps Into Remote, Inaccessible Areas
- Offers Quick, Long-Lasting Lubrication
- Penetrates To Loosen Seized & Corroded Metal Mechanisms
- Provides Free-Flowing Protection
- Provides Protection Against Rust & Corrosion

Applications

- Metal MechanismsMetal-To-Metal Surfaces
- Chain Drives
- Drag Lines
- Bushings
- Pulleys
- Hinges
- Tools
- Sleeve Bearings Open Gears
- Steel Cables
- Couplings
- Linkages
- Wheels
- Augers
- Rusty Nuts & Bolts
- Any Automotive, Industrial or Commercial Areas Of Lubrication That Require An External Heavy-Duty Spray Lubricant For Accessible And Hard-To-Reach Areas

Directions of Usage

· Spray to any frozen mechanisms

SPRAY SHIELD

SPRAY SHIELD[™] is the ultimate multi-purpose lubricant that also penetrates metal surfaces while maintaining highest gualities in corrosive and extreme humidity environments. SPRAY SHIELD[™] penetrates into remote areas and delivers long-lasting lubrication in many different applications. SPRAY SHIELD™ works quickly to provide excellent protection and long-lasting lubrication.



Properties	Standard	Unit	Result
Flash Point		°C	226
Boiling point		°C	238
Evaporation rate	0		<0.01
Vapor pressure			<1@25 °C
Specific gravity			1.07



I 5A. STRIKE-SHIELD, SPRAY-SHIELD & WD-40 COMPARISON

When comparing the characteristics of SPRAY-SHIELD (SS) & STRIKE-SHIELD (STKS) to those of WD-40, there are profound differences in product features and benefits.

WD-40 CLAIMS FIVE BASIC FUNCTIONS

also dissolves adhesives, allowing easy removal of labels, tape, dispersed and hold firmly to all moving parts. stickers, and excess bonding material.

DISPLACES MOISTURE: Because WD-40 displaces moisture, it quickly dries out electrical systems to eliminate moisture-induced short circuits.

CLEANS: WD-40 gets under dirt, grime and grease to clean. It LUBRICATES: WD-40's lubricating ingredients are widely

PROTECTS: WD-40 protects metal surfaces with corrosionresistant ingredients to shield against moisture and other corrosive elements

PENETRATES: WD-40 loosens rust-to-metal bonds and frees stuck. frozen or rusted metal parts.

SPRAY-SHIELD & STRIKE-SHIELD PROVIDE THE ABOVE FUNCTIONS PLUS MORE

CLEAN: SS & STKS lift and remove dirt, grime and grease from surfaces, including adhesive compounds and bonding agents.

SS & STKS keep surfaces clean by rejecting airborne contaminants, such as dust and smoke, due to its electrochemical surface bonding technique by causing a dipole-dipole interaction and cation exchange on the metal surfaces. This, in effect, creates a greater positive charge on the metal surfaces which react with positive charged airborne contaminants (+ ions) in causing a repulsion between the two.

DISPLACE MOISTURE: SS & STKS are a fast-acting drying agent for quick and thorough moisture displacement in damp or soaked electrical or electronic systems. A migrating film burrows under condensation and moisture, driving it to the surface where they dry or can be wiped off to reactivate circuits. After moisture is displaced, an ultra-thin residual film resists rust and corrosion. SS & STKS do not contain carbon tetrachloride and will not harm insulation.

Areas of Use: Wet or damp ignitions, electrical systems, motors, controls, starters, relays, radios, electronic equipment, etc.

PENETRATE: SS & STKS are the ideal solution to hundreds of different maintenance and production problems involving rust and corrosion. They work quickly and effectively on even the most severe cases of rusted equipment, parts and components. SS & STKS disperse quickly into rusted, corroded areas to loosen scale and free up working mechanisms, tight fitting parts and frozen fastenings. SS & STKS's low surface tension speed penetration even into normally inaccessible areas. SS & STKS will not affect painted surfaces.

Areas of Use: Rusted lugs or bolts, sliding parts, sluggish mechanisms, hinges, tools, products in storage, gear trains, wheels, rotating apparatus, linkages, cams, levers, industrial equipment. etc.

LUBRICATE: SS & STKS are a light, but lasting lubricant. They offer quick, positive, long-lasting lubrication on wide varieties



flowing and lubricating deeply into hard-to-reach internal areas.

slides, hinges, garden equipment, etc.

products' unique formulation uses a combination of the best rust

ADDITIONAL FEATURES OF SPRAY-SHIELD & STRIKE-SHIELD

SS & STKS contain Steel Shield EPA "ABF Technology"TM, one of misting of lubrication or pinpoint just the areas you want with jet the leading anti-friction, metal-treating products. EPA is a blend of spray. You can also use SS & STKS with Air Pressurized Sprayer, petroleum products containing no graphite, PTFE, silicones, moly which works as well as an aerosol, but utilizes compressed air. or synthetics. The dielectric strength of SS & STKS is rated at 45 KV (45,000 volts). Please see the attached page for a quick reference chart on the comparison of these products.

SS & STKS are not an aerosol - they are packaged in convenient hand pump sprav bottles and conforms to DOT and OSHA regulations. The adjustable nozzle allows you to provide an overall

Feature or Benefit	SS & STKS	WD-40
Loosens rusted parts	Within minutes	Within hours
Penetrates deep and fast	Immediately	Slowly
Keeps parts from freezing up	Long term	Short term
Protects metal against corrosion	Long term	Short term
Leaves a barrier film	Long term	Short term
Displaces moisture	YES	YES
Dries electrical equipment	YES	YES
Non-conductive	YES (45,000V DIELECTRIC)	YES (UNKNOWN)
Stops squeaks	Long term	Short term
Gummy residue	NONE	NONE
Long-lasting lubrication	YES	NO
Anti-wear properties	YES	NO
Extreme-pressure properties	YES	NO

of applications in industry, the shop and garage, the farm and and oxidation preventatives available. SS & STKS seek out rust, at home. SS & STKS are ideal for automobiles and equipment, stop the oxidation process and prevent further rust from forming.

In addition, SS & STKS remove corrosion and corrosion bridges Areas of Use: Metal mechanisms, gears, locks, linkages, wheel from printed circuit boards after a light spray and "toothbrush" bearings, metal-to-metal surfaces, cams, levers, engines, motors, scrubbing. This prohibits corrosion from also returning in the future. After corrosion is removed and excess SS & STKS are wiped from the board, a microfilm layer will remain on the surface of the metals PROTECT: SS & STKS stop rust dead in its tracks. These to virtually eliminate the long-term effects of electrolytic bridging.

TABLE I - SPRAY-SHIELD & STRIKE-SHIELD COMPARED TO WD-40

• Strike Shield and Spray Shield win

NOTE: The following disclaimer appeared on the WD-40 web site at the time of this writing: "The uses of WD-40 described on this Web site are provided to WD-40 Company by end-users of the product, and do not constitute recommendations or suggestions for use of WD-40 by WD-40 Company. These uses including the 'Use of the Day', have not been tested by WD-40 Company, Consumers should exercise common sense whenever using WD-40. Always follow the instructions and take heed of any warnings printed on the WD-40 packaging." WD-40 is a registered trademark of WD-40 Company, San Diego, California, U.S.A.

I 6. GREASE APPLICATIONS OF BEARINGS

I 6A. STEEL SHIELD GREASE COMPATIBILITY

I 6B. BEARING TYPES AND GREASES

COMMON APPLICATIONS OF

Cylindrical

DIFFERENT TYPES OF BEARINGS

Needle

	Re	CC	RA	•	•	•						
Polyurea (Shear Stable)	С	В	С	С	С	С	В	С	С	С	С	
Polyurea (Conventional)					С							С
Lithium Complex	С		С	С	С	С		С	С			С
Lithium 12-Hydroxy			В	С		В		С		С		С
Lithium Stearate			С	С		В			С	С		С
Clay (non-Soap)			С	С								В
Calcium Sulfonate Complex	В	С	С	В				В	В	С		С
Calcium Complex				В						С	С	С
Calcium 12-Hydroxy	С	С	С		В	В	С	С	С	С		С
Calcium Stearate				С		С	С	С	В	С		С
Barium Complex				С		С						В
Aluminum Complex				С		В				С		С
	uminum Complex	arlum Complex	alcium Stearate	alcium 12-Hydroxy	alcium Complex	aclium Sulfonate Complex	ay (Non-Soap)	thium Stearate	thium 12-Hydroxy	thium Complex	olyurea (Conventional)	olyurea (Shear Stable)
	Aluminum Co	Barlum Comp	Calcium Stea	Calcium 12-H	Calcium Com	Caclium Sulfo	Clay (Non-Soa	Lithium Stear	Lithium 12-Hy	Lithium Comp	Polyurea (Cor	Polyurea (She

ELATIVE OMPATIBILITY ATING

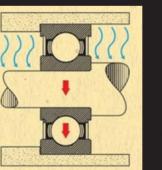
- B = Borderline
- C = Compatible
- I = Incompatible
- Note: This chart is a general guide to compatibility. Specific properties of greases can dictate compatibility. Testing
- should be done to determine if greases are compatible.



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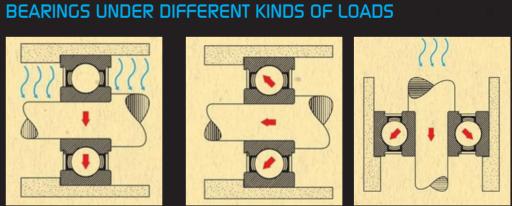


Radial Load When the load is perpendicular to the shaft due to gravity

Tapered

Axial or Thrust Load

gravity





BEARING TYPE AFFECTS GREASE LIFE

Bearing Type	Relative Type of Grease
Deep-groove, single-row ball bearing	
Angular contact, single-row ball bearing	0.625
Self-aligning, single-row bearing	0.77 - 0.625
Trust ball bearing	0.2 - 0.17
Cylindrical, single-row roller bearing	0.625 - 0.43
Needle roller bearing	0.3
Tapered roller bearing	0.3
Spherical roller bearing	0.14 - 0.08

Larger bearings and high-speed bearings translate to short grease life. High DN grease is required.

• When the load is parallel to the shaft • Left: Axial load in a horizontal pump • Right: Axial load in a vertical pump or electrical motor due to

17. HOW TO CHOOSE THE RIGHT GREASE

I 7A. SELECTION GUIDE

BASE OIL VISCOSITY

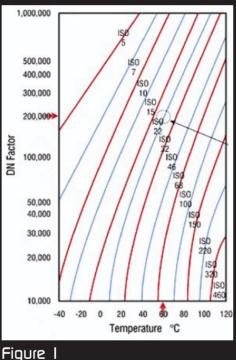
A common OEM grease specification might be to use an NLGI No. 2 lithium grease of good quality. Using this information alone, one could select the right consistency and thickener type. Other considerations include thickener concentration, consistency, dropping point and operating temperature range, worked stability, oxidation stability, wear resistance, etc.

A common mistake when selecting a grease is to confuse the grease consistency with the base oil viscosity. Because the majority of greaselubricated applications are element bearings, one should consider viscosity selection for those applications. While most would not use an EP 220 gear oil for an oil-lubricated electric motor bearing, many people will use a grease containing that same oil for an identical grease-lubricated bearing. To determine minimum and optimum viscosity requirements for element bearings, one may use speed factors, commonly denoted as DN or NDm. Speed factors account for the surface speed of the bearing elements and are determined by the following formulas:

The NDm value uses pitch diameter rather than bore diameter because not all bearings of a given bore have the same element diameter, and thus have different surface speeds. Knowing the speed factor value and likely operating temperature, the minimum viscosity requirement can be read directly from charts like Figure 1. Figure 1 assumes the base oils' viscosity index. To be more precise, one would need to use a chart that identifies the viscosity at operating temperature, then determine the viscosity grade from a viscosity / temperature chart for a given lubricant.

ADDITIVES AND BASE OIL TYPES

Figure 2 shows some common additive requirements by application. Most greases are formulated using API Group I and II mineral oil base stocks, which are appropriate for most applications. However, there are applications that might benefit from the use of a synthetic base oil. Such applications include high or low operating temperatures, a wide ambient temperature range, or any application where extended re-lubrication intervals are desired.



• The minimum viscosity requirement of oils



Additive	Journal Bearings	Ball Bearings	Thrust Bearings	Roller Bearings	Needle Bearings
Antioxidants	•	•	•	•	•
Antifoam Agents	•	•	•	•	•
Antiwear / EP		•	•	•	•
Rust Inhibitors	•	•	•	•	-
Extreme Pressure			-	-	
Demulsibility	•	•	•	•	-
VI Improvers	-	-	-	-	•
Corrosion Inhibitors	•	•	•	•	•

Figure 2

• Common additive requirements by applications

GREASE CONSISTENCY AND THICKENER TYPE

The NLGI has established a scale to indicate grease Like lithium complex, polyurea has good high-temperature consistency which ranges from grades 000 (semifluid) to 6 performance as well as high oxidation stability and bleed (block grease). The most common NLGI grade is two and resistance. Thickener type should be selected based on is recommended for most applications. performance requirements as well as compatibility when considering changing product types.

For bearings, speed factor and operating temperature can be used to determine the best consistency or NLGI grade for a given application. It may seem counterintuitive, but higher speed factors require higher consistency greases Table 1 provides a general guide to selecting NLGI grad based on speed factor and operating temperature.

Numerous types of grease thickeners are currently in use the most common types are simple lithium soaps, lithiu complex and polyurea. Simple lithium soaps are often use in general-purpose greases and perform relatively well most performance categories at moderate temperatures Complex greases such as lithium complex provide improved performance particularly at higher operating temperatures. A common upper operating temperature limit for a simple lithium grease might be 250°F, while that for a lithium complex grease might be 350°F. Another thickener type that is becoming more popular is polyurea.





Operating Temperature	DN (Speed Factor)	* NIGL No.
	0 - 75,000	1
-30 to 100 °F	75,000 - 150,000	2
	150,000 - 300,000	2
0	0 - 75,000	2
0 to 150 °F	75,000 - 150,000	2
	150,000 - 300,000	3
	0 - 75,000	2
100 to 275 °F	75,000 - 150,000	2
	150.000 - 300.000	3

* Depends on other factors as well, including bearing type, thickener type, base oil viscosity and base oil type



PERFORMANCE PROPERTIES

If an application operates continuously at room temperature, properties like dropping and upper operating temperature limits are not as important. If an application operates under heavy loads at low speeds, load carrying tests such as fourball EP or Timken OK load should be considered. It is also important to review these specifications on a periodic basis to guard against specification creep. While improving a lubrication program can be a tough job, lubricant specification is relatively easy. Armed with a little bit of knowledge and a few widely available tools, it is possible to rest easier knowing that the right grease is being used.

"With Steel Shield ABF Technology, the performance in stability, lubricity and interval of grease can be enhanced and improved to a much higher level."



NEVER USE GREASE OR OILS WITH SOLID ADDITIVES

"Solid additives such as Molybdenum will damage metal surface in the long term, never attempt to try those lubricants."

"Steel Shield lubricating oils and greases utilize unique ABF Technology which perform much better than any solid additives. Steel Shield is your wise choice."

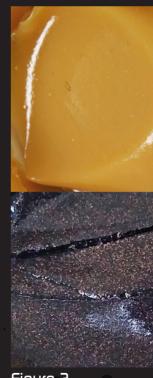


Figure 3
Upper: Steel Shield Grease - Lithi Shield does not contan any solid additive which have excellent performance in high loading operations without creating any undesire effects to metals
Lower: Grease contains solid additive (Molybdenum). It will damage metal surface in the long term because it contans solid materials





I 8. SOUTHWEST RESEARCH INSTITUTE TEST REPORTS

"Steel Shield lubricants with ABF Technology is proven to be superior to any other top-class lubricants in the world."



I 8A. GAS ENGINE OILS AND COMPRESSOR OILS TESTS

STEEL SHIELD REVEALED POWERFUL PERFORMANCE IN TESTS

			Products of t	he same class		
						
SwRI Lab No.	24564	23728	23252	23727	25250	25251
ASTM D2782 Measurement of Extreme-Pressure Properties of Lubricating Fluids (Timken Method)	SST Gas Engine Oil SAE 40 Ashless Without EPA	Steel Shield Gas Engine Oil GECAT SAE40 Low Ash With EPA	Steel Shield EPA	Steel Shield Compressor Oil ISO #100 / 150	Mobil Pegasus 805SAE 40 Gas Engine Oil	Mobil Pegasus 801SAE 40 Gas Engine Oil
Volume (Gallon)	1	1	1	1	1	1
OK Load (lbs)	40	40	75	55	9	9
Score Load (lbs)	45	45	80	60	12	12
Temperature (°C)	38	38	38	38	38	38

FIGURE I - STEEL SHIELD WINS IN Timken TEST (ASTM D2782)

• The SwRI Timken Test report clearly testified Steel Shield products are FAR Superior than Mobil products of the same classes

- Steel Shield outperforms Mobil in OK LOAD parameter by 444 % and $\,$ in SCORE LOAD by 375 %

			Products of t	he same class		
SwRI Lab No.	24564	23728	23252	23727	25250	25251
ASTM D2783 Measurement of Extreme-Pressure Properties of Lubricating Fluids (4-Ball Method)	SST Gas Engine Oil SAE 40 Ashless Without EPA	Steel Shield Gas Engine Oil GECAT SAE40 Low Ash With EPA	Steel Shield EPA	Steel Shield Compressor Oil ISO #100 / 150	Mobil Pegasus 805SAE 40 Gas Engine Oil	Mobil Pegasus 801SAE 40 Gas Engine Oil
Volume (Gallon)	1	4	1	1	1	1
Corrected Load (kgf)	70	109	NA	1	136	74
Load Wear Index (kgf)	35	46	NA	48	34	35
Weld Point (kg)	200	250	>800	250	200	200
Last Non Seizure Load (kg)	80	100	80	100	63	80

FIGURE 2 - STEEL SHIELD WINS IN 4-BALLS TEST (ASTM D2783)

• The SwRI 4-Balls Test testified Steel Shield products are superior than Mobil products of the same classes

• Steel Shield outperforms Mobil in the Weld Point (oil strength in resistant to EP) parameter by 129 % and in the Last Non Seizure Load (wear performance in respect to load) by 159 %.

• *** Remarks: 4-ball test is normally for heavy weight oil and grease.

FIGURE 3 - ORIGINAL TEST REPORTS FROM SWRI (RIGHT PAGE)

SOUTHWEST RESEARCH INSTITUTE°

November 20^h, 2014

George Fennell Steel Shield Technologies 3351 Industrial Blvd Bethel Park, PA 15102-2543 Phone: 1-800-390-1535 Email:

Re: Fuel Analysis Results SwRI W0# 71111 P0# 120

Dear Mr. Fennell:

Analyses have been completed on your samples in accordance with the tests requested. Twelve samples were received in good condition between July 21⁴, 2014 and October 7th 2014 in good condition. Eleven samples were received in one gollon plastic containers and one sample was received in a one quart plastic bottle. Sample Identification and testing requesting is shown in the table on the following page. Testing took place between October 13th and November 11th 2014. Test results and sample identifications are shown in the table attached.

Analyses were performed according to the listed ASTM test procedures with no modifications or deviations. Precision should be consistent with those stared in the ASTM test procedures. Sample aliquots were taken in accordance with the various ASTM test procedures. The analyses above pertain only to the sample received by Southwest Research Institute and represent only that sampling lot. This report shall not be reproduced except in full without the express written permission of Southwest Research Institute.

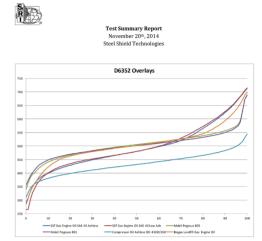
If there are any questions concerning these analyses, or if you need any additional testing on the samples, please contact me at (210) 522-2071. We appreciate the opportunity to be of service to your firm.



Benefiting government, industry and the public through innovative science and technolog

	<u>_</u> e		_					
				est Summary Re				
				November 20th, 20				
			Ste	eel Shield Technol	logies			
							SwRI Labi	<i>¥ 24</i>
COT 0								
5AE 40 A	Engine Oil							
	Plastic lug							
1 Gallon	i lastic jug							
ASTM D2	782 Measu	irement of	Extreme-	Pressure Propert	ies of Lubrica	ting Fluid	ls (Timken	Me
				·····				
S	core Load,	lbs						
Т	emperatur	е, °С						
				Pressure Propert				leth
L	ast won Sei	izure Load,	, кg					
				n of Petroleum Dis			. '	
IBP	285.3	20%	428.8	40% 464.8	60%	497.5	80%	
IBP 1%	285.3 306.2	20% 21%	428.8 431.1	40% 464.8 41% 466.4	60% 61%	497.5 499.2	80% 81%	57
IBP 1% 2%	285.3 306.2 333.2	20% 21% 22%	428.8 431.1 433.3	40% 464.8 41% 466.4 42% 467.9	60% 61% 62%	497.5 499.2 501.1	80% 81% 82%	57 57
IBP 1% 2% 3%	285.3 306.2 333.2 351.6	20% 21% 22% 23%	428.8 431.1 433.3 435.4	40% 464.8 41% 466.4 42% 467.9 43% 469.4	60% 61% 62% 63%	497.5 499.2 501.1 503.0	80% 81% 82% 83%	57 57 58
IBP 1% 2% 3% 4%	285.3 306.2 333.2 351.6 364.1	20% 21% 22% 23% 24%	428.8 431.1 433.3 435.4 437.2	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9	60% 61% 62% 63% 64%	497.5 499.2 501.1 503.0 505.0	80% 81% 82% 83% 84%	51 51 58 58
IBP 1% 2% 3% 4% 5%	285.3 306.2 333.2 351.6 364.1 373.5	20% 21% 22% 23% 24% 25%	428.8 431.1 433.3 435.4 437.2 439.2	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4	60% 61% 62% 63% 64% 65%	497.5 499.2 501.1 503.0 505.0 507.1	80% 81% 82% 83% 84% 85%	57 57 58 58 59
IBP 1% 2% 3% 4% 5% 6%	285.3 306.2 333.2 351.6 364.1 373.5 380.5	20% 21% 22% 23% 24% 25% 26%	428.8 431.1 433.3 435.4 437.2 439.2 441.2	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 474.0	60% 61% 63% 64% 65% 66%	497.5 499.2 501.1 503.0 505.0 507.1 509.3	80% 81% 82% 83% 84% 85% 86%	57 57 58 58 59 59
IBP 1% 2% 3% 4% 5% 6% 7%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 380.5 386.7	20% 21% 22% 23% 24% 25% 26% 27%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 443.1	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 474.0 47% 475.6	60% 61% 62% 63% 64% 65% 66% 67%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8	80% 81% 82% 83% 84% 85% 86% 87%	57 57 58 59 59 60
IBP 1% 2% 3% 5% 6% 7% 8%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 380.5 386.7 391.9	20% 21% 22% 23% 24% 25% 26% 27% 28%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 443.1 444.9	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 474.0 47% 475.6 48% 477.1	60% 61% 63% 64% 65% 66% 67% 68%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8 514.5	80% 81% 82% 83% 84% 85% 86% 87% 88%	51 51 58 59 60 60
IBP 1% 2% 3% 5% 6% 7% 8% 9%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 380.5 386.7 391.9 396.0	20% 21% 22% 23% 24% 25% 26% 27% 28% 29%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 443.1 444.9 446.7	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 477.0 47% 475.6 48% 477.1 49% 478.6	60% 61% 62% 63% 64% 65% 66% 67% 68% 69%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8 514.5 517.3	80% 81% 82% 83% 84% 85% 86% 87% 88% 89%	57 58 58 59 60 60 61
IBP 1% 2% 3% 5% 6% 7% 8% 9% 10%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 386.7 391.9 396.0 399.1	20% 21% 22% 23% 24% 25% 26% 26% 28% 28% 29% 30%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 443.1 444.9 446.7 448.6	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 474.0 47% 475.6 48% 477.1 49% 478.6 50% 480.2	60% 61% 63% 63% 65% 66% 67% 68% 69% 70%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8 514.5 517.3 520.4	80% 81% 82% 83% 84% 85% 86% 87% 88% 89% 90%	57 58 58 59 60 61 61 62
IBP 1% 2% 3% 4% 5% 6% 7% 8% 9% 10% 11%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 380.5 386.7 391.9 396.0	20% 21% 22% 23% 24% 25% 26% 26% 28% 28% 29% 30% 31%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 443.1 444.9 446.7	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 477.0 47% 475.6 48% 477.1 49% 478.6	60% 61% 62% 63% 64% 65% 66% 67% 68% 69%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8 514.5 517.3	80% 81% 82% 83% 84% 85% 86% 87% 88% 89%	57 58 59 60 61 62 63
IBP 1% 2% 3% 4% 5% 6% 7% 8% 9% 10% 11%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 386.7 391.9 396.0 399.1 403.0	20% 21% 22% 23% 25% 26% 27% 28% 29% 30% 31% 32%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 444.2 444.9 444.9 444.7 448.6 450.5	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 474.0 47% 475.6 48% 477.1 49% 478.6 50% 480.2 51% 481.8	60% 61% 63% 64% 65% 66% 67% 68% 69% 70% 71%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8 514.5 517.3 520.4 523.7	80% 81% 82% 83% 85% 86% 87% 88% 89% 90% 91%	57 57 58 59 60 60 61 62 63 63
IBP 1% 2% 3% 4% 5% 6% 7% 8% 9% 10% 11% 12%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 386.7 391.9 396.0 399.1 403.0 406.6	20% 21% 22% 23% 24% 25% 26% 26% 28% 29% 30% 31% 32% 33%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 444.9 444.9 444.7 444.6 450.5 452.1	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 474.0 47% 475.6 68% 477.1 49% 478.6 50% 480.2 51% 483.4 52% 483.4	60% 61% 62% 63% 65% 65% 66% 68% 69% 70% 71% 72%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8 514.5 517.3 520.4 523.7 527.3	80% 81% 82% 83% 85% 86% 87% 88% 88% 90% 91% 92%	57 57 58 59 60 61 62 63 63 64
IBP 1% 2% 3% 6% 6% 7% 8% 9% 10% 11% 12% 13% 14%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 386.7 391.9 396.0 399.1 403.0 406.6 410.2	20% 21% 22% 23% 24% 25% 26% 27% 26% 28% 29% 30% 31% 32% 33% 34%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 443.1 444.9 446.7 448.6 450.5 452.1 453.7	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 474.0 47% 475.6 48% 477.1 49% 478.6 50% 480.2 51% 481.8 52% 483.4 45% 485.1	60% 61% 62% 63% 65% 66% 67% 68% 69% 70% 71% 72% 73%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8 514.5 517.3 520.4 523.7 527.3 521.2	80% 81% 82% 83% 84% 85% 86% 87% 88% 89% 90% 91% 92% 93%	57 57 58 59 60 61 61 62 63 64 65
IBP 1% 2% 3% 5% 6% 7% 8% 9% 10% 11% 12% 13% 14%	285.3 306.2 333.2 351.6 364.1 373.5 380.7 391.9 396.0 399.1 403.0 406.6 410.2 413.5	20% 21% 22% 23% 24% 25% 26% 27% 28% 29% 31% 32% 33% 34% 35%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 443.1 444.9 444.7 444.6 450.7 452.1 453.7 455.2	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 474.0 47% 475.6 48% 477.1 49% 478.6 50% 482.4 51% 481.8 52% 483.4 53% 485.4 54% 486.8	60% 61% 63% 64% 65% 66% 68% 69% 70% 72% 72% 73% 74% 75%	497.5 499.2 501.0 505.0 507.1 509.3 514.5 514.5 517.3 520.4 523.7 527.3 531.2 531.2	80% 81% 83% 84% 85% 86% 86% 87% 88% 99% 90% 91% 92% 93% 93% 94%	57 57 58 59 60 61 61 61 61 61 61 61 61 61 61 61 61 61
IBP 1% 2% 3% 5% 6% 7% 8% 9% 10% 11% 12% 13% 14% 15% 16%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 386.7 391.9 396.0 399.1 403.0 406.6 410.2 413.5 416.5	20% 21% 22% 23% 24% 25% 26% 27% 28% 29% 30% 31% 32% 33% 34% 35% 36%	428.8 431.1 433.3 435.4 437.2 439.2 441.2 443.1 444.9 444.7 444.6 450.5 452.1 453.7 455.2 455.2 455.9	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.0 45% 472.4 46% 474.0 47% 475.6 50% 483.4 50% 483.4 53% 485.1 54% 485.4 53% 485.1 55% 488.5	60% 61% 63% 64% 65% 66% 68% 69% 70% 72% 72% 73% 74% 75%	497.5 499.2 501.0 505.0 507.1 509.3 511.3 514.5 517.3 520.4 522.7 527.3 531.2 535.3 539.6	80% 81% 82% 83% 84% 85% 86% 87% 88% 90% 91% 92% 91% 92% 93% 92%	57 57 58 59 60 61 62 63 64 65 66 66 60 67
IBP 1% 2% 3% 5% 6% 7% 8% 9% 10% 11% 12% 13% 14% 15% 16%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 386.7 391.9 396.0 399.1 403.0 406.6 410.2 413.5 416.5 419.1	20% 21% 22% 23% 24% 25% 26% 27% 28% 30% 31% 32% 33% 34% 35% 35% 37%	428.8 431.1 433.3 435.4 437.2 439.2 449.2 444.2 444.9 444.9 444.7 448.6 450.5 452.1 453.7 455.2 455.2 456.9 458.5	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.9 45% 472.4 46% 470.9 45% 472.4 46% 470.9 47% 475.6 50% 482.4 51% 481.8 52% 483.4 53% 485.1 54% 486.8 55% 490.2	60% 61% 62% 63% 65% 66% 66% 68% 68% 70% 71% 72% 72% 72% 72% 74% 73% 74% 75%	497.5 499.2 501.1 503.0 505.0 507.1 509.3 511.8 514.5 517.3 520.4 523.7 527.3 531.2 533.2 539.6 539.6	80% 81% 82% 83% 85% 86% 87% 88% 90% 91% 92% 93% 94% 95%	59 60 61 62 63 63 64 65 66 65 66 67 68
IBP 1% 2% 3% 4% 5% 6% 9% 10% 11% 12% 13% 14% 15% 16% 16%	285.3 306.2 333.2 351.6 364.1 373.5 380.5 380.5 386.7 391.9 396.0 399.1 403.0 406.6 410.2 413.5 410.2 413.5 419.1 421.8	20% 21% 22% 23% 24% 25% 26% 27% 28% 29% 30% 31% 32% 33% 34% 35% 36% 37% 38%	428.8 431.1 433.3 435.4 437.2 439.2 449.2 444.9 444.9 444.9 444.9 444.7 448.6 450.5 452.1 453.7 455.2 456.9 458.5 460.1	40% 464.8 41% 466.4 42% 467.9 43% 469.4 44% 470.0 45% 472.4 46% 474.0 47% 475.6 50% 487.1 49% 478.6 52% 483.4 53% 485.5 56% 490.2 57% 492.0	60% 61% 62% 64% 65% 66% 67% 68% 69% 70% 71% 72% 73% 74% 75% 76% 77%	497.5 499.2 503.0 505.0 507.1 509.3 511.8 514.5 517.3 520.4 523.7 522.3 531.2 535.3 539.6 544.2 549.2	80% 81% 82% 83% 85% 86% 87% 88% 89% 90% 91% 92% 93% 94% 94% 95% 97%	51 51 52 52 52 52 52 52 52 52 60 61 62 63 64 65 66 65 66 65 66 65 66

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In comparing the curves and D6352 chromatography, it is observed that samples STF Gas Engine oil SAE 40 Ashies and SST Gas Engine Oil SAE 40 Low Ash ar very similar with the exception that the Low Ash oil appears to have an added component that is somewahai lighter than the rest of the Oil. The build of this oil is lighter than the others, however it does have a larger proportion of heavier compounds. In general it has broader array of hydrocarbons than the other oils. The Moll Peaguus 803 and Moll Peaguus 805 are essentially the same oil with the same boiling distribution. They both are a narrower cut reducing the amount of lighter and heavier hydrocarbon species. The Bilogas Landfill Gas Engine Oil has a distribution in between the SST Gas Engine Oils and the Mobil Pegasus SOI and More Compressor oil is a significantly lighter oil than the rest of the samples.

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Test Summary Report November 20th, 2014 Steel Shield Technologies

1%	308.9	21%	465.1	41%	492.4	61%	514.0	81%	548.7
2%	331.8	22%	467.0	42%	493.5	62%	515.2	82%	552.3
3%	349.1	23%	468.8	43%	494.7	63%	516.5	83%	556.3
4%	362.7	24%	470.4	44%	495.8	64%	517.8	84%	560.5
5%	374.7	25%	472.0	45%	496.9	65%	519.1	85%	565.1
6%	385.9	26%	473.6	46%	497.9	66%	520.4	86%	569.9
7%	396.5	27%	475.1	47%	498.9	67%	521.8	87%	575.0
8%	406.2	28%	476.5	48%	499.9	68%	523.1	88%	580.8
9%	415.0	29%	477.8	49%	500.9	69%	524.5	89%	586.8
10%	422.4	30%	479.1	50%	502.0	70%	526.0	90%	593.2
11%	429.0	31%	480.4	51%	503.0	71%	527.5	91%	599.9
12%	434.9	32%	481.6	52%	504.0	72%	529.0	92%	607.5
13%	440.2	33%	482.9	53%	505.1	73%	530.7	93%	615.4
14%	444.7	34%	484.2	54%	506.1	74%	532.4	94%	624.3
15%	449.2	35%	485.4	55%	507.2	75%	534.2	95%	633.7
16%	452.5	36%	486.6	56%	508.2	76%	536.1	96%	644.5
17%	455.4	37%	487.8	57%	509.3	77%	538.1	97%	656.4
18%	458.3	38%	489.0	58%	510.5	78%	540.4	98%	671.9
19%	460.7	39%	490.1	59%	511.7	79%	542.8	99%	688.2
								FBP	697.9

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				est Summ November					
				November eel Shield T					
								SwRI Lab#	\$ 25252
SST-EPA									
1 Gallon	Plastic Jug								
ASTM D2	2782 Meas	urement of	Extreme-	Pressure P	roperties	of Lubrica	ting Fluids	s (Timken	Method)
0)kay Load,	lbs lbs							75 80
		re, °C							38
ASTM D2	2783 Meas	urement of .oad, kgf	Extreme-	Pressure P	roperties	of Lubrica	ting Fluids	s (4-Ball M	ethod)
L	oad Wear	Index, kgf							
L	Veld Point, .ast Non Se	, kg izure Load,	kg						>800 80
N									
individual or	the entity name	ntained in this do ed above. If the	reader of this	document is not	the intended	recinient, you a	re hereby notif	ied that any dis	semination.
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Note 2: Instit	tute shall not pu	ablish or make kr	nown to other	s the subject ma	tter or results	of the Project of	any informati	on obtained in	connection
to Institute or	r any of its emp	ry and confidenti bloyees, either dir	rectly or by in	nplication, shall	be made use a	of by Client or c	in Client's beha	alf without Inst	itute's
written appro used in its en	val. In the eve tirety, unless Ir	ent Client distribu astitute approves	a summary o	t issued by Insti r abridgement fo	tute on this Pr or distribution	oject outside its	own organiza	tion, such repor	t shall be
							ORRLA	KE4 Steel Sh	ield (a).docx age 15 of 16
SB									
SRI									
SR				'est Summa					
SR				' est Summ a November eel Shield T	20th, 2014	ł			
SR				November	20th, 2014	ł		SwRI Lab#	¢ 25250
	gasus			November	20th, 2014	ł		SwRI Lab#	ŧ 25250
805				November	20th, 2014	ł		SwR1 Lab#	± 25250
805 1 Gallon	Plastic Jug		Ste	November eel Shield T	20th, 2014 echnolog	les			
805 1 Gallon ASTM D2	Plastic Jug 2782 Meas	urement of	Extreme-	November eel Shield T Pressure P	20th, 2014 rechnolog	ies of Lubrica	ting Fluids	s (Timken	Method) 9
805 1 Gallon I ASTM D2 0 S	Plastic Jug 2782 Meas Okay Load, Icore Load,	urement of lbs	Ste	November eel Shield T Pressure P	20th, 2014 echnolog	l ies of Lubrica	ting Fluids	5 (Timken	Method) 9 12
805 1 Gallon I ASTM D2 0 S	Plastic Jug 2782 Meas Okay Load, Icore Load,	urement of lbs	Ste	November eel Shield T Pressure P	20th, 2014 echnolog	l ies of Lubrica	ting Fluids	5 (Timken	Method) 9
805 1 Gallon I ASTM D2 0 S T	Plastic Jug 2782 Meas 20kay Load, core Load, 'emperatu	urement of lbs lbs re, °C	Sta Extreme-	November eel Shield T Pressure P	20th, 2014 rechnolog	es of Lubrica	ting Fluid	s (Timken 	Method) 9 12 38
805 1 Gallon I ASTM D2 0 Si T ASTM D2 C	Plastic Jug 2782 Meas Okay Load, core Load, 'emperatur 2783 Meas corrected L	urement of lbs lbs re, °C urement of .oad, kgf	Extreme-	November eel Shield T Pressure P Pressure P	20th, 2014 rechnolog	es of Lubrica	ting Fluids	s (Timken s (4-Ball M 	Method) 9 12 38 ethod) 136
805 1 Gallon I ASTM D2 0 S T T ASTM D2 C L	Plastic Jug 2782 Meas Okay Load, core Load, 'emperatur 2783 Meas corrected L	urement of lbs lbs re, °C urement of .oad, kgf Index, kgf	Extreme-	November eel Shield T Pressure P Pressure P	20th, 2014 echnolog roperties roperties	es of Lubrica	ting Fluids	s (Timken s (4-Ball M 	Method) 9 12 38 ethod)
805 1 Gallon I O S T ASTM D2 C C L W	Plastic Jug 2782 Meas Okay Load, core Load, 'emperatur 2783 Meas corrected L oad Wear Veld Point,	urement of lbs lbs re, °C urement of .oad, kgf Index, kgf	Sta Extreme-	November eel Shield T Pressure P Pressure P	20th, 2014 echnolog roperties roperties	es of Lubrica	ting Fluids	s (Timken s (4-Ball M 	Method) 9 12 38 ethod) 136 34
805 1 Gallon I ASTM D2 O S S T ASTM D2 C C L W L U U L	Plastic Jug 2782 Meas 3kay Load, core Load, "emperatu 2783 Meas Corrected L oad Wear Veld Point, ast Non Se	urement of lbs lbs lbs urement of .oad, kgf Index, kgf kg izure Load,	Extreme-	November eel Shield T Pressure P Pressure P	20 th , 2014 echnolog roperties	of Lubrica	ting Fluids	s (Timken s (4-Ball M 	Method) 9 12 38 ethod) 136 34 200
805 1 Gallon I ASTM D2 O S S T ASTM D2 C C L W L U U L	Plastic Jug 2782 Meas 3kay Load, core Load, "emperatu 2783 Meas Corrected L oad Wear Veld Point, ast Non Se	urement of lbs lbs re, °C urement of .oad, kgf Index, kgf	Extreme-	November eel Shield T Pressure P Pressure P	20 th , 2014 echnolog roperties	of Lubrica	ting Fluids	s (Timken s (4-Ball M 	Method) 9 12 38 ethod) 136 34 200
805 1 Gallon I ASTM D2 0 S T ASTM D2 C L W L ASTM D6 IBP	Plastic Jug 2782 Meas Jkay Load, core Load, 'emperatui 2783 Meas corrected L .oad Wear Veld Point, ast Non Se 5352 Boilir 338.1	urement of lbs lbs re, °C urement of .oad, kgf Index, kgf kg izure Load, ng Range Dis 20%	Extreme- Extreme- kg	November eel Shield T Pressure P Pressure P n of Petrole 40%	20th, 2014 echnolog roperties roperties um Distill 495.3	of Lubrica of Lubrica lates from	ting Fluids ting Fluids 174 to 700 515.0	s (Timken s (4-Ball M) °C by GC 80%	Method) 9 12 38 ethod) 136 34 200 63 538.2
805 1 Gallon I ASTM D2 0 S T ASTM D2 C L W L W L	Plastic Jug 2782 Meas 0kay Load, core Load, remperatu 2783 Meas corrected L oad Wear Veld Point, ast Non Se 5352 Boilir	urement of lbs re, °C urement of .oad, kgf Index, kgf , kg izure Load,	Extreme-	November eel Shield T Pressure P Pressure P	20 th , 2014 echnolog roperties roperties um Distill	of Lubrica of Lubrica	ting Fluids	s (Timken s (4-Ball M s o °C by GC	Method) 9 12 38 ethod) 136 34 200 63
805 1 Gallon I ASTM D2 O S T ASTM D2 C L U U L ASTM D6 IBP 1% 2% 3%	Plastic Jug 2782 Meas 2782 Meas 2783 Meas corrected L cord Vead 2783 Meas corrected L cord Vear Veld Point, ast Non Se 3352 Boilir 338.1 363.1 384.2 396.2	urement of lbs lbs re, °C Index, kg index, kg izure Load, sg Range Die 20% 21% 22% 23%	Extreme- Extreme- kg	November eel Shield T Pressure P Pressure P n of Petrole 40% 41% 42% 43%	roperties roperties um Distill 495.3 496.4 497.4 498.3	of Lubrica of Lubrica dates from 60% 61% 63%	ting Fluids ting Fluids 174 to 700 515.0 516.1 517.1 518.1	s (Timken s (4-Ball M) °C by GC 80% 81% 83% 83%	Method) 9 12 38 ethod) 136 34 200 63 538.2 539.6 541.0 542.6
805 1 Gallon I ASTM D2 O S T ASTM D2 C C L W W L ASTM D6 IBP 1% 2% 3% 4% 5%	Plastic Jug 2782 Meas Dkay Load, core Load, 'emperatur 2783 Meas corrected L corrected L c	urement of lbs	Extreme- Extreme- kg	November eel Shield T Pressure P Pressure P n of Petrole 40% 41% 42% 43% 44%	20th, 2014 echnolog roperties um Distill 495.3 496.4 497.4 497.4 498.3 499.3 500.3	of Lubrica of Lubrica of Lubrica ates from 60% 61% 62% 63% 64% 65%	ting Fluids ting Fluids 174 to 700 515.0 515.1 517.1 518.1 518.1 519.2 520.3	s (Timken s (4-Ball M) °C by GC 80% 81% 82% 83% 83% 84%	Method) 9 12 38 ethod) 136 34 200 63 538.2 539.6 541.0 542.6 544.2 545.9
805 1 Gallon I ASTM D2 O S. T ASTM D2 C C L W L ASTM D6 IBP 1% 2% 3% 3% 4% 5%	Plastic Jug 2782 Meas Kay Load, core Load, emperatu 2783 Meas corrected L coad Wear Veld Point, ast Non Se 3352 Boilir 338.1 363.1 384.2 396.2 401.9 410.8 419.2	urement of lbs	Extreme- Extreme- kg	November eel Shield T -Pressure P 	20%, 2014 roperties roperties um Distill 495.3 495.4 497.4 497.4 499.3 500.3 501.3	of Lubrica of Lubrica of Lubrica ates from 61% 62% 63% 64% 65% 66%	ting Fluids ting Fluids 174 to 700 515.0 516.1 517.1 518.1 518.1 518.1 519.2 520.3 521.4	s (Timken 	Method) 9 12 38 ethod) 136 34 200 63 538.2 539.6 541.0 542.6 544.2 545.9 545.9
805 1 Gallon 1 ASTM D2 S ASTM D2 C C L W W L ASTM D6 IBP 1% 2% 3% 4% 5% 6% 7% 8%	Plastic Jug 2782 Meas kay Load, core Load, core Load, core Load, emperatui 2783 Meas corrected L cond Wear Veld Point, ast Non Se 3352 Boilin 338.1 363.1 384.2 396.2 401.9 410.8 419.2 426.0 431.6	urement of lbs	Extreme- Extreme- kg	November eel Shield T 	20%,2014 roperties roperties um Distill 495.3 496.4 497.4 497.4 497.4 499.3 501.3 501.3 501.3 502.2	of Lubrica of Lubrica of Lubrica ates from 61% 62% 63% 64% 65% 66% 66% 66% 66% 66%	ting Fluids ting Fluids 174 to 700 515.0 516.1 518.1 518.1 518.1 518.1 518.1 518.1 518.1 518.1 518.1 518.1 518.1 518.1 512.3 522.6	s (Timken s (4-Ball M) °C by GC 81% 81% 82% 83% 84% 84% 85% 86% 86% 87% 88%	Method) 9 12 38 ethod) 136 34 200 63 538.2 539.6 541.0 542.6 544.2 545.9 547.7 549.7 551.8
805 1 Gallon 1 ASTM D2 O S T ASTM D2 C L U V L ASTM D6 IBP 1% 2% 3% 4% 5% 6% 6% 8% 9%	Plastic Jug 2782 Meas 3084 Load, core Load,	urement of lbs	Stu Extreme- Extreme- kg	November eel Shield T 	20% 2014 roperties roperties um Distill 495.3 496.4 497.4 498.3 501.3 501.3 502.2 503.2	of Lubrica of Lubrica of Lubrica lates from 60% 61% 63% 64% 65% 66% 66% 66% 66% 66% 66% 66% 66%	ting Fluids ting Fluids 174 to 700 515.0 516.1 517.1 519.2 520.3 521.4 522.5 523.6 524.7	s (Timken s (4-Ball M) °C by GC 81% 81% 81% 83% 83% 83% 85% 86% 86% 86% 86% 86% 86% 86% 88% 88% 88	Method) 9 12 38 ethod) 136 34 200 63 538.2 539.6 541.0 542.6 541.0 542.6 542.5 549.7 549.7 554.1
805 1 Gallon 1 ASTM D2 S. ASTM D2 C C L W L ASTM D6 BP 1% 2% 3% 6% 6% 6% 6% 9% 10% 11%	Plastic Jug 2782 Meas kkay Load, core Load, 'emperatu 2783 Meas corrected L coad Wear Veld Point, ast No Se 3352 Boilir 338.1 363.1 384.2 396.2 401.9 410.8 419.2 426.0 431.6 436.1	urement of lbs	Extreme- Extreme- kg	November cel Shield T Pressure P Pressure P Pressure P 40% 41% 42% 43% 44% 44% 45% 43% 45% 50% 51%	20%-2014 roperties roperties um Distill 495.3 496.4 497.4 498.3 501.3 501.3 501.2 503.2 504.1 505.1	k ies of Lubrica of Lubrica dates from 60% 61% 62% 63% 64% 65% 66% 65% 66% 67% 66% 70% 71%	ting Fluid: ting Fluid: 174 to 700 515.0 519.2 520.3 521.4 522.5 523.6 524.7 525.8	s (Timken 	Method) 9 12 38 ethod) 136 34 200 63 538.2 539.6 541.0 542.6 542.6 542.4 542.5 549.7 549.7 554.1 554.1 555.5
805 1 Gallon I ASTM D2 C C C C C C C C C W W L ASTM D2 C C C C C C C C C C C C C	Plastic Jug 2782 Meas 2782 Meas 2783 Meas 2784	urement of lbs	Extreme- Extreme- kg	November eel Shield T Pressure P Pressure P	20%-2014 roperties roperties 495.3 495.4 495.3 495.4 495.3 501.3 501.3 501.2 505.1 506.9	4 ees of Lubrica of Lubrica ates from 60% 61% 63% 64% 65% 66% 67% 66% 67% 67% 67% 67% 67%	ting Fluids ting Fluids 515.0 516.1 517.1 518.1 519.2 520.3 521.4 522.5 523.6 524.7 525.8	s (Timken 	Method) 9 12 38 ethod) 136 63 538.2 539.6 541.0 542.6 544.2 544.2 544.2 544.2 544.2 544.5 545.1 551.8 556.8 556.8
805 1 Gallon I ASTM D2 C C C C C C C C C C C C C	Plastic Jug 782 Meas Nay Load, core Load, emperatu 2783 Meas 783 Meas	urement of lbs	State Stribution Stributio Stribution Stribution Stribution Stribution Stribution Stribu	November eel Shield T 	20%-2014 roperties roperties 495.3 495.4 495.3 495.4 497.4 498.3 501.2 503.2 503.2 503.2 503.2 503.2 503.2 505.1 506.9 507.9	4 eres of Lubrica of Lubrica dates from dates from date	ting Fluid: ting Fluid: 174 to 700 515.0 516.1 517.1 518.1 518.1 518.2 52.6 524.5 526.5 526.5 526.5 526.5 526.5 526.1 529.3 528.1 529.3		Method) 9 12 38 4 4 200 63 538.2 539.6 541.0 542.6 544.5 544.5 544.5 554.8 555.8 555
805 1 Gallon 1 ASTM D2 C C C C C C C C C C C C C	Plastic Jug 2782 Meas kay Load, core Load, 'emperatu 7783 Meas Ourrected L oad Wear Veld Point, ast Non Se 3352 Boilin 338.1 363.1 384.2 396.2 401.9 410.8 419.2 426.0 431.6 436.1 440.5 444.1	urement of lbs	State Extreme- Extreme- kg	November eel Shield T 	20%-2014 roperties roperties um Distill 495.3 496.4 497.4 497.4 497.4 497.3 501.3 501.3 501.3 501.3 504.1 505.1 506.0 506.9	4 ecs of Lubrica of Lubrica ates from 60% 64% 66% 66% 66% 66% 66% 66% 66% 66% 67% 66% 66	ting Fluid: ting Fluid: 174 to 700 515.0 516.1 519.2 520.3 521.4 522.5 524.7 525.6 524.7 525.6 524.9 526.9 528.1	s (Timken	Method) 9 12 33 8 ethod) 136 34 200 63 538.2 539.6 542.6 544.9 542.6 544.9 542.6 544.9 542.7 549.7 554.1 554.1 554.1 555.1 8 555.1
805 1 Gallon 1 ASTM D2 C C C C C C C C C C C C C	Plastic Jug 2782 Meass 2783 Meass 2783 Meass 2783 Meass 2783 Meass 2783 Meass 2783 Meass 2783 Meass 2783 Meass 2784 Meass 2783 Meass 2784	urement of lbs	Extreme- Extreme- kg	November eel Shield T Pressure P Pressure P	um Distill 495.3 496.4 497.4 496.4 497.4 496.4 497.4 496.4 497.4 496.4 497.4 496.4 497.4 496.4 497.4 496.4 496.4 497.4 503.5 504.1 505.1 506.0 506.0 506.0 506.9 507.9 508.9 509.9 511.9	e les of Lubrica of Lubrica of Lubrica ates from ates from ates from 60% 63% 64% 64% 64% 64% 64% 64% 64% 64	ting Fluid: ting Fluid: 174 to 700 515.0 517.1 519.2 523.6 524.7 525.8 524.7 525.8 524.7 525.8 524.7 525.8 529.3 530.5 531.7 533.0	s (Timken 	Method) 9 38 ethod) 136 34 200 63 34 200 63 538.2 539.6 541.0 554.5 554.5 554.9 554.1 556.5 554.9 554.9 554.9 554.9 554.9 554.9 558.9 568.0 569.0 568.0 568.0 568.0 569.0 568.0 569.0 569.0 568.0 569.
ASTM D2 O S T ASTM D2 C L W W L X ASTM D6 BIP 1% 6 3% 6 7% 6 8% 6 7% 6 8% 6 7% 6 8% 12% 9% 12% 13% 6 13% 6 13% 6 13% 6 13% 6 13% 6 13% 7 10 10 10 10 10 10 10 10 10 10 10 10 10	Plastic Jug (792 Meass kay Load, (2783 Meass (2783 Meass (2783 Meass (2783 Meass (2783 Meass (2783 Meass (2783 Meass (2783 Meass (2783 Meass) (2783	urement of lbs	Extreme- Extreme- kg	November eel Shield T Pressure P Pressure P Pressure P Pressure P (1) (1) (1) (1) (1) (1) (1) (1)	20%-2014 echnolog roperties roperties um Distill 495.3 497.4 496.4 497.4 496.4 497.4 496.3 500.3 500.3 500.3 500.3 505.1 506.9 507.9 508.9 508.9 508.9	4 etes of Lubrica of Lubrica dates from tates from 60% 63% 63% 63% 63% 63% 63% 63% 63	ting Fluid: ting Fluid: 515.0 517.4 to 700 517.1 518.1 518.1 518.1 518.2 528.3 528.5 528.5 528.5 528.1 528.3 528.1 529.3 528.1 529.3 531.7		Method) 9 12 38 ethod) 136 334 200 63 538.2 239.6 63 539.6 539.6 539.6 539.5 542.0 542.0 554.1 554.2 554.2 554.3 555.5 556.5 556.3 556.2 556.3 556.2 5

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Test Summary Report November 20th, 2014 Steel Shield Technologies

	SwRI Lab# 23727
Compressor Oil Ashless	
SO #100/150	
1 Gallon Plastic Jug	
ASTM D2782 Measurement of Extreme-Pressure Properties of Lubricating Fluid	ls (Timken Method)
Okay Load, lbs	55
Score Load, lbs	60
Temperature, °C	38

re Properties of Lubricating Fluids (4-Ball Method) D2783 Measurement of Extro Corrected Load, kgf...... Load Wear Index, kgf..... Weld Point, kg... Last Non Seizure Load, kg.. 250

ASTM D6352 Boiling Range Distribution of Petroleum Distillates from 174 to 700 °C by GC

IBP	310.0	20%	398.6	40%	421.7	60%	442.0	80%	465.5	
1%	326.9	21%	400.0	41%	422.7	61%	443.1	81%	466.9	
2%	344.5	22%	401.4	42%	423.6	62%	444.1	82%	468.4	
3%	354.0	23%	402.7	43%	424.6	63%	445.3	83%	469.9	
4%	360.6	24%	404.0	44%	425.6	64%	446.4	84%	471.5	
5%	365.4	25%	405.2	45%	426.6	65%	447.5	85%	473.2	
6%	369.2	26%	406.4	46%	427.6	66%	448.7	86%	474.9	
7%	372.5	27%	407.7	47%	428.6	67%	449.8	87%	476.7	
8%	375.5	28%	408.9	48%	429.6	68%	450.9	88%	478.7	
9%	378.2	29%	410.1	49%	430.6	69%	452.0	89%	480.7	
10%	380.6	30%	411.2	50%	431.6	70%	453.1	90%	483.0	
11%	382.8	31%	412.4	51%	432.6	71%	454.2	91%	485.6	
12%	384.9	32%	413.4	52%	433.6	72%	455.4	92%	488.3	
13%	386.9	33%	414.5	53%	434.6	73%	456.6	93%	491.4	
14%	388.9	34%	415.5	54%	435.7	74%	457.8	94%	494.9	
15%	390.7	35%	416.6	55%	436.7	75%	459.0	95%	498.8	
16%	392.4	36%	417.7	56%	437.7	76%	460.2	96%	503.3	
17%	394.0	37%	418.7	57%	438.8	77%	461.5	97%	509.1	
18%	395.6	38%	419.7	58%	439.9	78%	462.8	98%	517.6	
19%	397.1	39%	420.7	59%	440.9	79%	464.1	99%	531.3	
								FBP	544.3	

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					ary Repor				
					20th, 2014				
			Ste	el Shield T	echnologi	es			
								SwRI Lab#	2525
Mobil Pe	tasus								
801	, ,								
1 Gallon	Plastic Jug								
ASTM D2	782 Measu	rement o	Extreme-F	Pressure P	roperties	of Lubrica	ting Fluid	ls (Timken I	Metho
	kay Load, II								
S	core Load, I	bs							1
Т	emperature	e, °C							3
	783 Measu orrected Lo							ls (4-Ball M	ethod
	oad Wear I								3
	eld Point,								20
L	ast Non Sei:	zure Load	, kg						8
ASTM D6	352 Boiling	g Range D	istribution	of Petrole	um Distill	ates from	174 to 70	0 °C by GC	
IBP	355.5	20%	469.5	40%	492.3	60%	510.0	80%	532
	372.7	21%	470.9						
1%				41%	493.3	61%	511.0	81%	533
2%	391.1	22%	472.3	42%	494.3	62%	511.9	82%	533 535
2% 3%	401.9	22% 23%	472.3 473.7	42% 43%	494.3 495.2	62% 63%	511.9 512.9	82% 83%	533 535 536
2% 3% 4%	401.9 413.3	22% 23% 24%	472.3 473.7 475.0	42% 43% 44%	494.3 495.2 496.2	62% 63% 64%	511.9 512.9 513.9	82% 83% 84%	533 535 536 538
2% 3% 4% 5%	401.9 413.3 422.1	22% 23% 24% 25%	472.3 473.7 475.0 476.2	42% 43% 44% 45%	494.3 495.2 496.2 497.0	62% 63% 64% 65%	511.9 512.9 513.9 514.9	82% 83% 84% 85%	533 539 530 538 539
2% 3% 4% 5% 6%	401.9 413.3 422.1 429.3	22% 23% 24% 25% 26%	472.3 473.7 475.0 476.2 477.4	42% 43% 44% 45% 46%	494.3 495.2 496.2 497.0 497.8	62% 63% 64% 65% 66%	511.9 512.9 513.9 514.9 516.0	82% 83% 84% 85% 86%	533 539 536 538 538 539 541
2% 3% 4% 5% 6% 7%	401.9 413.3 422.1 429.3 435.4	22% 23% 24% 25% 26% 27%	472.3 473.7 475.0 476.2 477.4 478.5	42% 43% 44% 45% 46% 47%	494.3 495.2 496.2 497.0 497.8 498.7	62% 63% 64% 65% 66% 67%	511.9 512.9 513.9 514.9 516.0 517.0	82% 83% 84% 85% 86% 87%	533 539 538 538 539 541 543
2% 3% 4% 5% 6% 7% 8%	401.9 413.3 422.1 429.3 435.4 440.6	22% 23% 24% 25% 26% 27% 28%	472.3 473.7 475.0 476.2 477.4 478.5 479.7	42% 43% 44% 45% 46% 47% 48%	494.3 495.2 496.2 497.0 497.8 498.7 499.5	62% 63% 64% 65% 66% 67% 68%	511.9 512.9 513.9 514.9 516.0 517.0 518.0	82% 83% 84% 85% 86% 87% 88%	533 538 538 538 538 538 541 543 543
2% 3% 4% 5% 6% 7%	401.9 413.3 422.1 429.3 435.4	22% 23% 24% 25% 26% 27%	472.3 473.7 475.0 476.2 477.4 478.5	42% 43% 44% 45% 46% 47%	494.3 495.2 496.2 497.0 497.8 498.7	62% 63% 64% 65% 66% 67%	511.9 512.9 513.9 514.9 516.0 517.0	82% 83% 84% 85% 86% 87%	533 536 536 538 539 541 543 545 545 547
2% 3% 4% 5% 6% 7% 8% 9%	401.9 413.3 422.1 429.3 435.4 440.6 444.6	22% 23% 24% 25% 26% 27% 28% 29%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8	42% 43% 44% 45% 46% 47% 48% 49%	494.3 495.2 496.2 497.0 497.8 498.7 499.5 500.4	62% 63% 65% 66% 67% 68% 69%	511.9 512.9 513.9 514.9 516.0 517.0 518.0 519.1	82% 83% 84% 85% 86% 87% 88% 89%	533 538 538 538 539 541 543 543 543 543 543
2% 3% 5% 6% 7% 8% 9% 10%	401.9 413.3 422.1 429.3 435.4 440.6 444.6 448.3	22% 23% 25% 26% 27% 28% 29% 30%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8 481.9	42% 43% 44% 45% 46% 47% 48% 49% 50%	494.3 495.2 496.2 497.0 497.8 498.7 499.5 500.4 501.2	62% 63% 65% 66% 67% 68% 69% 70%	511.9 512.9 513.9 514.9 516.0 517.0 518.0 519.1 520.2	82% 83% 84% 85% 86% 87% 88% 89% 90%	533 538 538 538 538 543 543 543 543 543 543 543 543
2% 3% 5% 6% 7% 8% 9% 10% 11%	401.9 413.3 422.1 429.3 435.4 440.6 444.6 444.3 451.6	22% 23% 25% 26% 27% 28% 29% 30% 31%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8 481.9 483.1	42% 43% 45% 46% 47% 48% 49% 50% 51%	494.3 495.2 496.2 497.0 497.8 498.7 499.5 500.4 501.2 502.1	62% 63% 64% 65% 66% 67% 68% 69% 70% 71%	511.9 512.9 513.9 514.9 516.0 517.0 518.0 519.1 520.2 521.3	82% 83% 84% 85% 86% 87% 88% 89% 90% 91%	533 538 538 538 541 543 543 543 543 543 543 543 543 543 543
2% 3% 5% 6% 7% 8% 9% 10% 11% 12%	401.9 413.3 422.1 429.3 435.4 440.6 444.6 444.3 451.6 454.2	22% 23% 24% 25% 26% 27% 28% 29% 30% 31% 32%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8 481.9 483.1 484.2	42% 43% 44% 45% 46% 47% 48% 49% 50% 51% 52%	494.3 495.2 496.2 497.0 497.8 498.7 499.5 500.4 501.2 502.1 503.0	62% 63% 65% 66% 67% 68% 69% 70% 71% 72%	511.9 512.9 513.9 514.9 516.0 517.0 518.0 519.1 520.2 521.3 522.4	82% 83% 84% 85% 86% 87% 88% 89% 90% 91% 92%	533 536 538 539 541 543 545 545 549 552 559
2% 3% 5% 6% 7% 8% 9% 10% 11% 12% 13%	401.9 413.3 422.1 429.3 435.4 440.6 444.6 444.3 451.6 454.2 456.7	22% 23% 24% 25% 26% 27% 28% 29% 30% 31% 32% 33%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8 481.9 483.1 484.2 485.2	42% 43% 44% 45% 46% 47% 48% 49% 50% 51% 52% 53%	494.3 495.2 496.2 497.0 497.8 498.7 499.5 500.4 501.2 502.1 503.0 503.8	62% 63% 64% 65% 66% 68% 69% 70% 71% 72% 73%	511.9 512.9 513.9 514.9 516.0 517.0 518.0 519.1 520.2 521.3 522.4 523.5	82% 83% 84% 85% 86% 87% 88% 90% 91% 92% 93%	533 536 538 541 543 545 547 549 552 559 559 563
2% 3% 5% 6% 7% 8% 10% 11% 12% 13% 14%	401.9 413.3 422.1 429.3 435.4 440.6 444.6 444.6 444.3 451.6 454.2 456.7 459.0	22% 23% 24% 25% 26% 29% 30% 31% 32% 33% 34%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8 481.9 483.1 484.2 485.2 486.3	42% 43% 44% 45% 46% 47% 48% 50% 51% 52% 53% 54%	494.3 495.2 496.2 497.0 497.8 498.7 499.5 500.4 501.2 502.1 503.0 503.8 504.7	62% 63% 65% 66% 66% 69% 70% 71% 72% 73% 74%	511.9 512.9 513.9 514.9 516.0 517.0 518.0 519.1 520.2 521.3 522.4 523.5 524.7	82% 83% 84% 85% 86% 87% 88% 89% 90% 91% 92% 93% 94%	533 535 536 538 539 541 543 545 547 549 552 555 559 563 568 575
2% 3% 4% 5% 6% 7% 8% 9% 10% 11% 12% 13% 14%	401.9 413.3 422.1 429.3 435.4 440.6 444.6 444.3 451.6 454.2 456.7 459.0 461.0	22% 23% 24% 25% 26% 28% 29% 30% 31% 32% 33% 34% 35% 36% 37%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8 481.9 483.1 484.2 485.2 486.3 487.3	42% 43% 44% 46% 47% 48% 50% 51% 52% 53% 54% 55% 56% 57%	494.3 495.2 496.2 497.0 497.8 498.7 499.5 500.4 501.2 502.1 503.0 503.8 504.7 505.5	62% 63% 64% 65% 66% 69% 70% 71% 72% 73% 74% 75% 76% 76%	511.9 512.9 513.9 514.9 516.0 519.1 520.2 521.3 522.4 523.5 524.7 525.9 527.1 528.3	82% 83% 85% 86% 87% 88% 89% 90% 91% 92% 93% 94% 95% 96% 97%	533 536 538 539 541 543 545 547 549 552 559 563 568
2% 3% 4% 5% 6% 9% 10% 11% 12% 13% 14% 15% 16% 16% 18%	401.9 413.3 422.1 429.3 435.4 440.6 444.6 444.3 451.6 454.2 456.7 459.0 465.7 459.0 464.7 466.5	22% 23% 24% 25% 26% 29% 30% 31% 32% 33% 34% 35% 36% 37% 38%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8 481.9 483.1 484.2 485.2 486.3 487.3 488.4 489.4 490.3	42% 43% 44% 46% 47% 48% 50% 51% 52% 53% 54% 55% 56% 57% 58%	494.3 495.2 496.2 497.0 497.8 498.7 499.5 500.4 501.2 502.1 503.0 503.8 504.7 505.5 506.4 507.2 508.1	62% 63% 65% 66% 67% 68% 69% 71% 72% 73% 74% 75% 76% 75% 76% 77% 68% 78%	511.9 512.9 513.9 514.0 517.0 518.0 519.1 520.3 522.4 522.5 522.4 523.5 522.4 523.5 522.4 525.9 527.1 528.3 529.6	82% 83% 85% 85% 86% 87% 89% 90% 91% 92% 92% 92% 94% 95% 96% 97% 97%	533 535 536 538 539 541 543 545 547 549 552 559 563 568 575 560 633
2% 3% 4% 5% 6% 7% 8% 9% 10% 11% 12% 13% 14% 15% 16% 17%	401.9 413.3 422.1 429.3 435.4 440.6 444.6 444.3 451.6 454.2 456.7 459.0 461.0 462.9 464.7	22% 23% 24% 25% 26% 28% 29% 30% 31% 32% 33% 34% 35% 36% 37%	472.3 473.7 475.0 476.2 477.4 478.5 479.7 480.8 481.9 483.1 484.2 485.2 485.2 485.2 486.3 487.3 488.4 488.4	42% 43% 44% 46% 47% 48% 50% 51% 52% 53% 54% 55% 56% 57%	494.3 495.2 496.2 497.8 497.8 499.5 500.4 501.2 502.1 502.1 503.8 504.7 505.5 506.4 507.2	62% 63% 64% 65% 66% 69% 70% 71% 72% 73% 74% 75% 76% 76%	511.9 512.9 513.9 514.9 516.0 519.1 520.2 521.3 522.4 523.5 524.7 525.9 527.1 528.3	82% 83% 85% 86% 87% 88% 89% 90% 91% 92% 93% 94% 95% 96% 97%	533 539 539 539 542 543 544 552 559 559 560 559 560 579 590

I 8B. STEEL SHIELD VS. YAMAMOTO AND ATLAS

STEEL SHIELD LARGELY OUTPERFORMS REPUTED GREASES MADE BY YAMAMOTO AND ATLAS

SwRI	Sample ID:		20003	20004	
Code:	Sample Identification:		Litho Shield	Yamamoto EP grease	
D1264	Water Washout of Grease				
	Avg. Grease Washed Out	Wt%	1.32	0.66	
	Test Temp.	°C	79	79	
	Dry Temp.	*C	77	77	
D1742	Oil Separation from Lubricating Grease	mass %	2.04	* Note	
D2265	Dropping Point	*C	258	307	
	Oven Temp.	*C	288	316	
D2266	Wear Characteristics (Four-Ball Method)				
	Scar Diameter	kgf	0.75	0.47	
D2596	Four-Ball Extreme Pressure Properties				
	Corrected Load	kgf	851.1	501.68	
	Load-Wear Index	kgf	92.27	66.73	
	Weld Point	kgf	800	315	
	LNSL	kgf	80	63	

sample "Yamatomo EP grease", therefore, the scope of the method

intended only for the use of the v notified that any dissemination, ediately notify us by telephone at propretary business informa the Intended recipient, you are th ad this document in error, please ass via the United States Postal S ubject matter or results of the Project or a

E)	OMRROJY 68350.65 Pege 2 e
	Benafiling government, inclusivy and the public through increasive science and lechoology

Test Items	Four-Ball Extreme Pressure Properties	Steel Shield Lithi Shield	Yamamoto EP Grease	Atlas Chisel Lube
Loading Ability	Corrected Load	851.1	501.68	302.79
Anti-Wear Ability	Load Wear Index	92.27	66.73	41.23
High Temperature Loading	Weld Point	800	315	315
High Pressure Loading	LNSL	80	63	50

Figure I

Grease test report summary

 Steel Shield Lithi Shield is superior to Yamamoto EP Grease and Atlas Chisel Lube in Loading ability, Anti-Wear Ability, High Temperature Loading and High Pressure Loading.

teel Shield Technolog Purchase Order # 11

SwRI	Sample ID:		20005
Code:	Sample Identification:		Atlas Chisel lube
D1264	Water Washout of Grease		
	Avg. Grease Washed Out	Wt %	1.11
	Test Temp.	*C	79
	Dry Temp.	*C	77
D1742	Oil Separation from Lubricating Grease	mass %	** Note
D2265	Dropping Point	*C	302
	Oven Temp.	*C	316
D2266	Wear Characteristics (Four-Ball Method)		
	Scar Diameter	kgf	0.71
D2596	Four-Ball Extreme Pressure Properties		
	Corrected Load	kgf	302.79
	Load-Wear Index	kgf	41.23
	Weld Point	kgf	315
	LNSL	kgf	50

** No oil separation occurred for grease sample "Atlas Chisel Lube", therefore, sample is considered "outside the scope of the method".

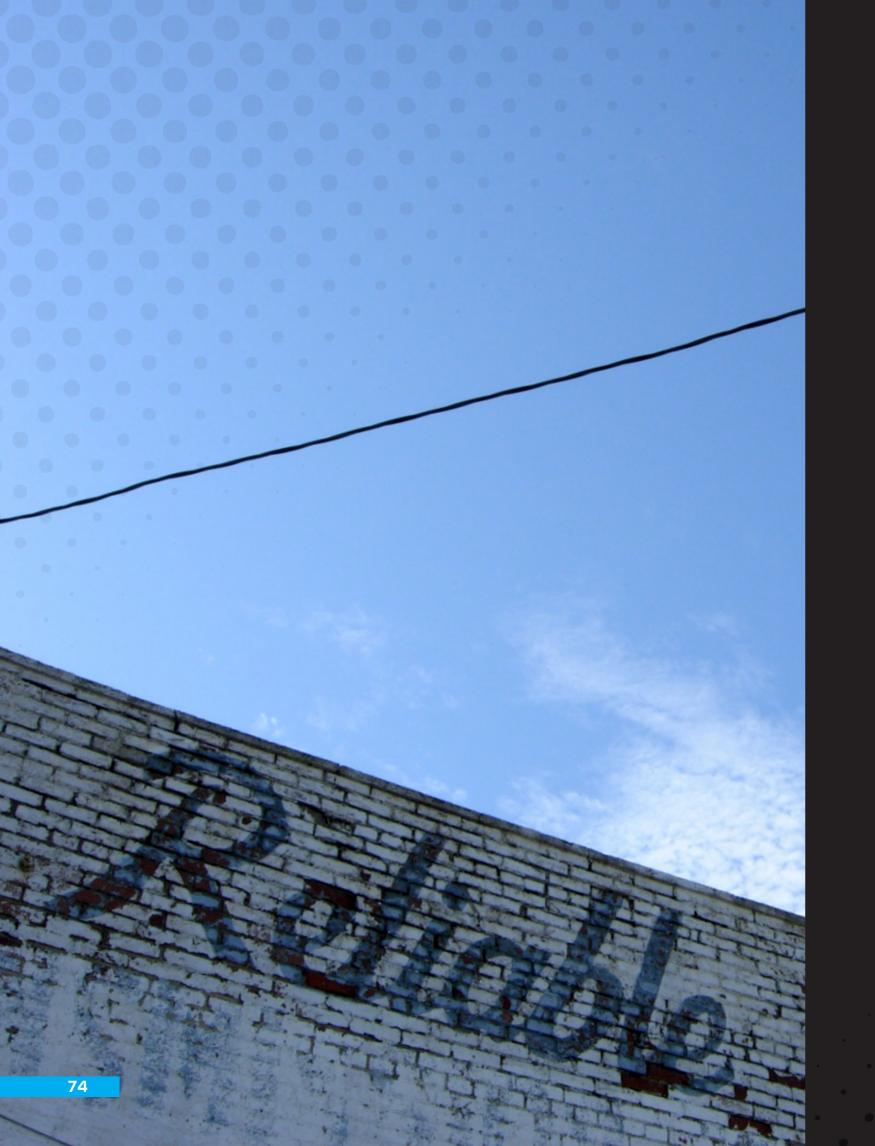
Not 11 This Individual of this decourse is apply privately and privately applicable privately power to prevail privately only for the use of the decourse of the second s to others the subject matter or results of the Project or

CNRRDJY 69250.docx Page 3 of 3

Benafiling government, industry and the public through



Figure 2 • Steel Shield (right) and Atlas (left)



I 9. INSURANCE CERTIFICATE & CONFIRMATION OF NO INSURANCE CLAIM

"The most powerful and reliable evidence showing that Steel Shield is the only lubricant you can trust."



I 9A. CERTIFICATE OF LIABILITY INSURANCE

OVER USD 2.000.000 INSURANCE

ACORD [®] CERTIF	ICATE OF LIA	BILITY IN	ISUR4	NCE		E (MM/DD/YYYY)
THIS CERTIFICATE IS ISSUED AS A MATT CERTIFICATE DOES NOT AFFIRMATIVELY BELOW. THIS CERTIFICATE OF INSURAN REPRESENTATIVE OR PRODUCER, AND TH	ER OF INFORMATION ONL OR NEGATIVELY AMEND ICE DOES NOT CONSTITU	LY AND CONFERS	NO RIGHTS	UPON THE CERTIFICA	TE HO	E POLICIES
IMPORTANT: If the certificate holder is an the terms and conditions of the policy, certi certificate holder in lieu of such endorseme	ADDITIONAL INSURED, the ain policies may require an					
PRODUCER	nt(s).	CONTACT Jamie	McDonald			
Best Insurance Agency		PHONE (724		FAX (A/C No)	(724)	283-1160
340 S. Main St., P.O. Box 670		E-MAIL ADDRESS: jamie@	bestinsur	ancebutler.com		
				RDING COVERAGE		NAIC #
Butler PA 16003	-0670		nnati In	surance Compani	es	
Steel Shield Technologies Inc		INSURER B : INSURER C :			-	
3351 Industrial Blvd /		INSURER D :				
		INSURER E :				
Bethel Park PA 15102		INSURER F :				
THIS IS TO CERTIFY THAT THE POLICIES OF II INDICATED. NOTWITHSTANDING ANY REQUIR CERTIFICATE MAY BE ISSUED OR MAY PERT. EXCLUSIONS AND CONDITIONS OF SUCH POLIC	EMENT, TERM OR CONDITION AIN, THE INSURANCE AFFORI CIES, LIMITS SHOWN MAY HAV	AVE BEEN ISSUED TO N OF ANY CONTRACT DED BY THE POLICIE TE BEEN REDUCED BY	OR OTHER S DESCRIBE PAID CLAIMS	DOCUMENT WITH RESPE D HEREIN IS SUBJECT T	CT TO	WHICH THIS
NSR TYPE OF INSURANCE ADDL	SUBR POLICY NUMBER	POLICY EFF (MM/DD/YYYY)	POLICY EXP (MM/DD/YYYY)	LIM	TS	
GENERAL LIABILITY				EACH OCCURRENCE DAMAGE TO RENTED	5	1,000,00
X COMMERCIAL GENERAL LIABILITY A CLAIMS-MADE X OCCUR	ENP04242014	4/24/2014	4/24/2015	PREMISES (Ea occurrence) MED EXP (Any one person)	s	100,00
				PERSONAL & ADV INJURY	5	1,000,00
				GENERAL AGGREGATE	\$	2,000,00
GEN'L AGGREGATE LIMIT APPLIES PER:	and the second sec			PRODUCTS - COMPIOP AGG		2,000,00
X POLICY PRO- JECT LOC				COMBINED SINGLE LIMIT	\$	
ANY AUTO				(Ea accident) BODILY INJURY (Per person)	\$	
ALL OWNED SCHEDULED AUTOS NON-OWNED				BODILY INJURY (Per accident)	\$	
HIRED AUTOS				PROPERTY DAMAGE (Per accident)	\$	
X UMBRELLA LIAB OCCUR				5400 0000 0000 000	\$	1,000,00
A EXCESS LIAB CLAIMS-MADE	ENP04242014	4/24/2014	4/24/2015	EACH OCCURRENCE AGGREGATE	5	1,000,00
DED RETENTION \$			-		s	
WORKERS COMPENSATION AND EMPLOYERS' LIABILITY Y/N			3	TORY LIMITS OTH		
ANY PROPRIETOR/PARTNER/EXECUTIVE				E.L. EACH ACCIDENT	\$	
(Mandatory in NH) If yes, describe under DESCRIPTION OF OPERATIONS below				E.L. DISEASE - EA EMPLOYE E.L. DISEASE - POLICY LIMIT		
Descramon or OPERATIONS DESK				EL DISENSE POLICE DMIT		
DESCRIPTION OF OPERATIONS / LOCATIONS / VEHICLES (A						
CERTIFICATE HOLDER		CANCELLATION				
Steel Shield Technologies 3351 Industrial Blvd. Bethel Park, PA 15102	Inc.	SHOULD ANY OF THE EXPIRATIO ACCORDANCE W	N DATE TH	DESCRIBED POLICIES BE C EREOF, NOTICE WILL CY PROVISIONS.	BE D	LLED BEFORE ELIVERED IN
ACORD 25 (2010/05)		(pm	wm	ORD CORPORATION.	_	

FIGURE I - ORIGINAL CERTIFICATE OF LIABILITY INSURANCE

Best Insurance Agency

September 18, 2013

Steel Shield Technologies (Asia Pacific) Limited 22nd Floor, W. Business Centre 4 Kam Hong Street North Point, Hong Kong

To Whom It May Concern:

Please be advised that Steel Sheild Technologies Inc, manufacturer of specialty lubricants and greases, located in Bethel Park,Pennsylvania, USA, has had no claims, claim related incidents or notices of loss under any General Liability coverage continously since April 24, 2008

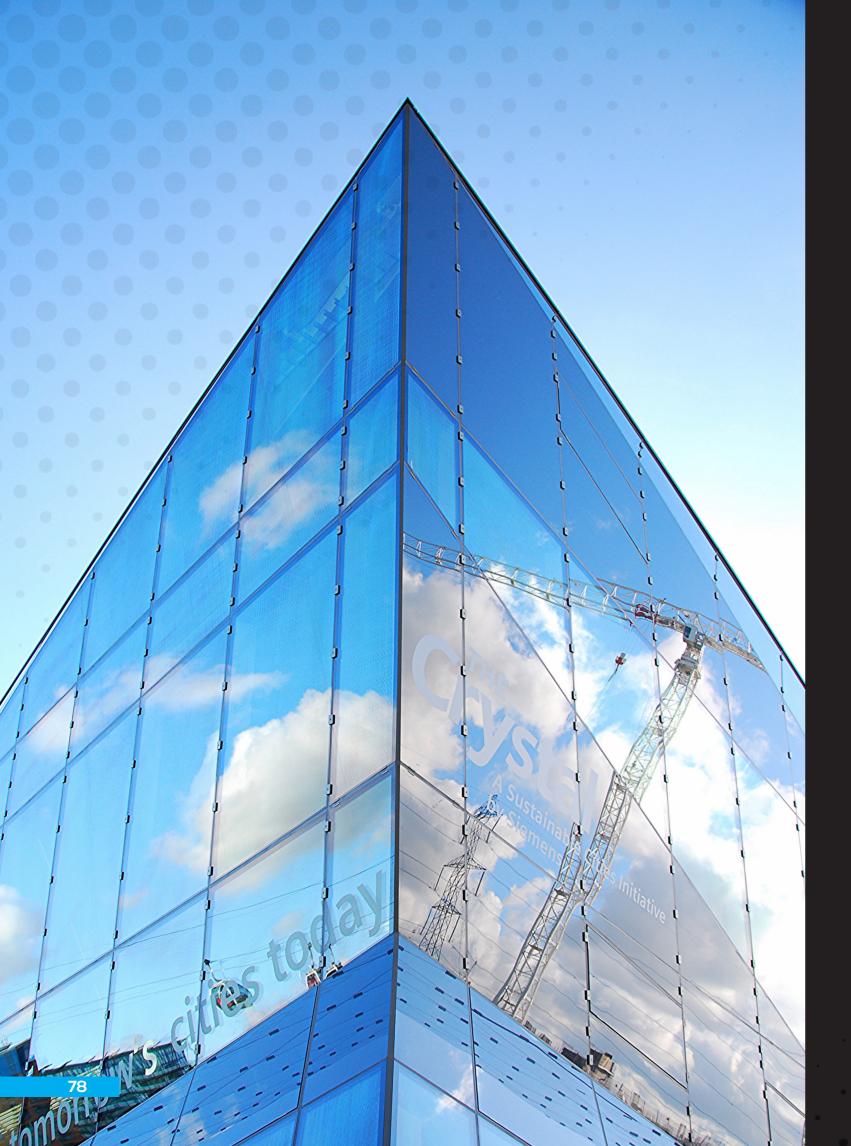
If you have any questions or need further information please feel free to contact me. I will be happy to be of further assistance.

Sincerely, Raymond A. Rosenbauer Vice President

FIGURE 2 - CONFIRMATION LETTER OF NO INSURANCE CLAIM Steel Shield had NEVER been claimed

INSURANCE

340 S. Main St., P.O. Box 670 Butler, PA 16003-0670 (724)283-5670 (724)283-1160Fax Email: Ray@Bestinsurancebutler.com



20. AUTHORITATIVE COMPLIMENTS LETTERS

"Formalized respectful action paid to STEEL SHIELD by the most renowned cooperations in the world. IN STEEL SHIELD, WE TRUST!"

20A. COMPLIMENTS FROM THE US ARMED FORCES

WEAPON SHIELD WAS TRUELY A LIFE SAVER



07 May 2008

Mark W. Pushnick sident & CEO Steel Shield Technologies, Inc 3351 Industrial Blvd Bethel Park, PA 15102-2543

Mark

I wanted to take time to express my sincere thanks to you and Steel Shield Technologies, Inc. for your support while I was deployed overseas in support of the Global War on Terrorism.

Your product, Weapon Shield, was truly a "life saver".

In my first combat tour to Afghanistan in late 2003, not knowing much about your product, I began to use it for my personal weapon and my crew-served vehicle weapon as a just another oil that I received in my care packages from home. I soon became educated on how this product was head and shoulders above the rest.

In the grueling conditions of southwestern Afghanistan, our weapons were subject to severe heat, dust, and even potential rust due to the humidity in the area. Compared to the other oils that we Stars, and seen potential row one to the humany in the area. Compared to the other offs that we received, Weapon Shield was the only product that stood up to the battlefield environment and did not cause the bolt of the weapons to become "gummy" or "sticky". <u>Weapon Shield actually acted as a "shield" and as a dust repellent.</u>

When I found out that I was deploying back to Iraq in 2007, one of my first calls was to my father to get my hands on Weapon Shield. While conducting pre-deployment training at Fort Bragg. I introduced my soldiers to this product. When it comes to selling to a tough audience, young enlisted men are some of the toughest to buy into a new idea. Within days, all of the men were carrying this product and were even hoarding bottles within their packs.

When we got to Iraq, Weapon Shield bottles became a part of the combat packing list as assigned with the gotto may reappend the other of the standing operating Procedure, a small of the on each man and tube of grease in each truck.

Weapons Shield brought us through over 25 fire fights with great success when other soldier's from different unit's weapons failed. On one occasion on patrol with another unit, their .50 cal machine gun jammed. One of my gunners tossed a bottle of Weapon Shield to them. They broke down their weapon, applied the shield and quickly got back into the firefight. In our mission after iction review, my soldiers quickly commented on how their weapons would only be treated with his product.

The bottom line is this... In two combat tours to both Afghanistan and Iraq, weapons treated with Weapon Shield, NEVER jammed. That saved lives. As a unit commander, my most important job was to complete this mission while bringing all of my soldiers home. Weapon Shield was a great contributor to my unit accomplishing that mission. In combat, the only option is perfect. If you are not, you can die. Weapon Shield was PERFECT every time. Victory!

Craig A. Hickerson MAJOR, Infantry USAR

FIGURE I (ABOVE) - US SOLIDERS HOLDING STEEL SHIELD BANNER

-



FIGURE 2 (RIGHT) - EMBLEM OF THE UNITED STATES DEPARTMENT OF THE ARMY

FIGURE 3 (LEFT) - LETTERS OF THANKS AND COMPLIMENTS FROM THE US ARMY

20B. COMPLIMENTS FROM THE SIEMENS

STEEL SHIELD PRODUCTS ARE GREAT CONTRIBUTORS TO SIEMEN'S SUCCESS



December 10, 2008

Mark W. Pushnick President & CEO Steel Shield Technologies, Inc. 3351 Industrial Blvd.

Mark

have used in the past.

We are currently using the Lithi-Shield grease in our shop and it has proven to work very well in our high temperature applications. We have experienced absolutely no down time due to bearing failure on our high temp furnace since we began using the Lithi-Shield grease. In the past all bearings were replaced on a quarterly basis causing a significant amount of downtime and material cost. We also use the grease in our automated welding equipment and anywhere else frequent greasing is needed. It has out performed our previously used grease in every application and we use it as often as possible.

Because of the performance of the Lithi-Shield grease we started using Steel Shield EPA in all of our metalworking equipment. Since its introduction to our machines we have not experienced a significant breakdown of any kind and it has left them running smoother and quieter than ever. The Steel Shield Drill and Tap fluid is also used our shop and has significantly decreased our tooling costs and become a favorite of most of our machinists. The Spray Shield product is used by our maintenance department and it is proving to be superior to anything used here in the past. We are very happy with the cost and performance of Steel Shield Technologies products and I highly recommend them. I am continually looking for ways to reduce costs and downtime Steel Shield products have been a great contributor to our success.

Bob Cavill Maintenance Department Supervisor Siemens VAI Services, LLC 2901 Industrial Blvd. Bethel Park, PA 15102 412-851-6700

FIGURE I - THE ORIGINAL LETTER OF COMPLIMENT FROM SIEMENS



I would like to take this opportunity to thank you for introducing us to Steel Shield Technologies line of lubricants and Metal Treatment products. The performance of your products has been overwhelmingly superior to any other lubricants or metal treatments we



UNION PACIFIC RAILROAD USES STEEL SHIELD PRODUCT EXTENSIVELY. STEEL SHIELD HAS BEEN PROVED TO BE FUNCTIONAL AND COST-EFFECTIVE. AND ARE HIGHLY RECOMMENDED



UNION PACIFIC

May 5, 2008

· Martin

351 Indi Park, PA

Dear Mr. Pushnick

d like to take a moment to endorse your products. I have been involved with your Advanced lary Film Technology for quite a few years on various Regions of the Union Pacific Railroad with m instructions per each Region. On the regions that have used your products continuously, they have reat results with the products experiencing increased longevity of the equipment and reduced down usly, they have

The last year and half I have been without the use of your products and have experienced increased failures an all areas of our equipment maintenance program (Engines, Transmissions, Hydraulic Systems, Gearboxes and Rear-ends).

I recently got the commitment of the Service Unit Director of Track (Kevin Hicks) to put out instructions for the next twelve months, the only grease that is to be used on the equipment is the LTHI-ISHIELD Grease. It will be doing my part, which is supplying the Rail Equipment Shield (Meal Tratament) for the rest of the applications as needed. I have great expectations of getting back the results that I experiment in the past over the next twelve months. Hopefully this will be the last plic program needed to prove the ROI hat these products provide with increased longevity of the equipment and reduced down time.



FIGURE I - THE ORIGINAL LETTER FROM MR. RICHARD E. BEST, THE MANAGER OF THE WORK EQUIPMENT

6455 East Commerce Cansas City, MO 64120

.

Marla Carrov 6455 East Co as City, **RE: MT-10**

Marla

10/01/200

I want to update you on our progress with the MT-10 product. Sense my last report I have applied MT10 to all of my service units. We use the product in our engines, transmissions, gearboxes and hydraulic tanks thus protesting the entire systems. The product performed as expected. Our failures with these components have decreased even more. Now we are able to work on equipment from the preventative maintenance side ased ever ad of a breakdown mode.

We have had cases that I can attribute directly to MT10 and were able to save the company thousands of dollars on the spot. This product proves itself worthy over and over and should be used by all departments to get the maximum savings for the Union Pacific.

Since	arely			
Man	Hendricks ager M/W Equ 245-2733	ipment Operations CR	Ł	
	-			
		÷		
Joe F. Hend	Iricks upment Operations			

FIGURE 2 - THE ORIGINAL LETTER FROM MR. JOE HENDRICKS, THE MANAGER M/W EQUIPMENT OPERATIONS CR

20D. LETTERS FROM THE PA PORT AUTHORITY

SAVED AROUND USD 45 IN MAINTENANCE COST FOR EVERY USD ONE DOLLAR INVESTMENT IN STEEL SHIELD PRODUCTS. ALSO, THE VEHICLES MALFUNCTIONS DROP TO AROUND 10%

FIGURE I (TOP RIGHT) - THE ORIGINAL LETTER FROM MR. MARK P. FERRARI, C.P.M., A.P.P., MANAGER OF CONTRACT ADMINISTRATION OF BUS & RAIL

FIGURE 2 (BOTTOM LEFT) - THE GEARBOX SYSTEM OF LIGHT RAILS

FIGURE 3 (BOTTOM RIGHT) - THE LIGHT **RAIL VEHICLES OF PORT AUTHORITY**





PORT AUTHORITY

Mark Pushnick President Mark Pushnick Enterprises 3351 Industrial Blvd. Bethel Park, PA 15102-2543

Re: Return on Investment of MT-10 Metal Treatment

Dear Mr. Pushnick:

As you are probably aware, Port Authority of Allegheny County's experiences with MT-10, has been very good. We have been using this product in the gearboxes of our light rail vehicles for approximately 8 of the last 9 years now. One year we discontinued the use of MT-10 and experienced a sharp decline in gearbox reliability and since have resumed the use of its application

August 14, 2002

We regularly have oil analysis performed, by an independent testing laboratory and the results of the analysis have indicated that the use of MT-10 has significantly lowered the wear metals that we previously experienced prior to its use. The MT-10 has appreciably extended the service life of our existing gearboxes.

Based on the costs we were incurring prior to the use of the MT-10 product verse the costs we are currently incurring, we have realized a Return On Investment (ROI) of approximately \$45 saved for every \$1 expensed or 45:1 ratio. The most significant factor was the increase in reliability as well as availability. The vehicles were able to perform when needed and the missed trips were lowered to approximately 10% of past history.

As you are also aware, we continue to use the Power Cut (PC-10) and Power Lift (PL-10) grease with similar

If you have any questions or I can be of any further assistance, feel free to contact me at (412) 566-5149.

Sincerely. Mad & Fernin

Mark P. Ferrari, C.P.M., A.P.P. Manager of Contract Admin Bus & Rail

mial is neither a solicited or paid testin

20E. LETTER FROM VOLVO CHINA

.

THE HORSEPOWER OF VOLVO VEHICLES INCREASED BY 8% TO 12% AFTER USING STEEL SHIELD PRODUCTS



FIGURE I - THE ORIGINAL LETTER FROM VOLVO

Volvo Car Corporation

8th November, 2013

To: Steel Shield Technologies (Asia Pacific) Limited Unit K, 11/F, Leader Industrial Centre, Phase 2, 188-202 Texaco Road, Tsuen Wan, H.K.

Dear Ms. Eva Lam.

We would like to express our gratitude to Steel Shield Technologies (Asia Pacific) Limited for providing Steel Shield lubricants for our vehicle horse power tests. In this test, our company applied Steel Shield lubricants to 4 Volvo cars (model: Volvo S80). We mounted the 4 cars on the horse power testing machines (Dyno-Shaft On-Vehicle Dynamometer) and conducted the tests individually.

The results indicate that, the 4 Volvo cars which had Steel Shield lubricants applied got horse power boosted by 8% - 12% compared with the same 4 cars without Steel Shield lubricants. Our company will be pleased to recommend Steel Shield to our customers.

Volvo Car Corporation R/M 1613, 2th Phase, Tongce Square, 3688 Jiangnan Road, Binjiang, Hangzhou, China Tel.: 0571-86852031 www.sinoworldcars.com

FIGURE 2 - THE ENGLISH TRANSLATION FROM THE ORIGINAL LETTER OF VOLVO



FIGURE 3 - VOLVO V70







2I. STEEL SHIELD HVAC CASE STUDIES

"Concrete feedbacks and appreciations from customers are important for the success of STEEL SHIELD."



1987 - AETOS CONSTRUCTION COMPANY

In 18th May, 1987, 4 oz. Steel Shield EPA was added to the crankcase of a Dunham / Bush 30 HP 4-cylinder refrigeration compressor based on an estimated system oil capacity of 5 gallons (Table 1). In 29th June, 1987, on the recommendation of George Fennel (the inventor and owner of Steel Shield), the compressor crankcase was drained, and refilled with new 4-GS oil and 16 oz. of Steel Shield EPA (Table 2).

Compared with the initial readings before adding Steel Shield EPA, the compressor showed a consistent 5% reduction in current draw with Steel Shield EPA. This reduction in current draw alone would result in an energy savings of USD 285 per year per compressor.



FIGURE I - DUNHAM / BUSH 30 HP 4-CYLINDER REFRIGERATION COMPRESSOR SYSTEM

Date	Ambient Temperature	Suction Pressure	Discharge Pressure	L1 Amps	L2 Amps	L3 Amps
5/18	80°F	23	120	44	45	48
5/19	74°F	23	110	44	44	48
5/20	72°F	23	110	43	43	47
5/21	74°F	24	115	44.5	46	48.3

Date	Ambient Temperature	Suction Pressure	Discharge Pressure	L1 Amps	L2 Amps	L3 Amps
6/29	76°F	23	115	44	46	46
6/30	76°F	23	115	42	44	44
7/01	78°F	23	118	42	43	44

Table I - COMPRESSOR READINGS ON

5/18/87 - 5/21/87 Added 40 oz. of Steel Shield EPA to the crankcase of the compressor based on an estimated system oil capacity of 5 gallons

Table 2- COMPRESSOR READINGS ON

6/29/87-7/1/87

 On the recommendation of George Fennel, the compressor crankcase was drained, and refilled with new 4-GS oil and 16 oz. of Steel Shield EPA

1989 - GEORGIA-PACIFIC

In the morning of 30th June, 1986, Steel Shield EPA was added to a A Fuller 1000 Vane Compressor. After one hour, the operating current dropped from 7.4 amps (normal condition) to 6.4 amps which reduced energy consumption by 14%.

FIGURE 2 - FULLER I 000 VANE COMPRESSOR

TABLE 3 (BELOW) - TEST LOGS ON 6/30/89

Equipment:	Fuller 1000 Vane Compressor	
Details:	No previous heat problems	
Before:	7.4 amps average consumption	
Activity:	6/30/89 7:00 a.m.	Added Ste
	6/30/89 8:00 a.m.	Average c

1989 - JOHN BEATTY REFRIGERATION & HEATING

CASE 1

System Configurations: a 10 ft x 20 ft walk-in freezer; R-502 refrigerant; using a Copeland MRA-0500 compressor, 3-phase, approx. 14 years old; Box temperature approx. -10°F.

Problem & Solution: "One of two condenser fans had quit. Compressor, in high ambient conditions, locked up. By reversing directions several times, we were able to break compressor loose, but unit was drawing almost full lockedrotor current. Oil level OK. As a what-do-we-have-to-lose, was applied, and the motor speed picked up as amperage steadily dropped. This was in the summer of 1986 and unite



el Shield EPA

nsumption 6.4 amps, 14% reduction



FIGURE 3 - COPELAND MRA-0500 COMPRESSOR



CASE 2

System Configurations: Hill 8 ft Frozen Meat case, upright; Copeland LAM-0310, 3-phase, semi-hermetic compressor; R-502 refrigerant; normal case temp. -5°F to -15°F; June 1989.

Problem & Solution: "Service call on case-high case temp. Case iced up. Case de-iced by over-night shut-down. Restarted next day; refrigerant and oil levels checked and adjusted if necessary. Compressor noisy. After 2 days, case still would not pull below +20. On third day, Steel Shield EPA was added to crankcase; nothing else was donw (or could be, short of changing out compressor). Customer reported next day that case was below zero. System continues to operate

FIGURE 4 - COPELAND LAM-0310 COMPRESSOR

CASE 3

System Configurations: 8 ft Produce case; Copeland KAJ-0100, 1-phase; R-12 refrigerant; Case temp. +35°F to +38°F; March 1989.

Problem & Solution: "Compressor would run for only a few seconds before going out on overload, due to being mechanically "tight". Oil level low, but within specs. Steel Shield EPA added. Again, system operating normally."

CASE 4

System Configurations: 8 ft x 8 ft walk-in Cooler; R-12 refrigerant; Tecumseh ¾ hp. hermetic, 1-phase; Temperature approx. +35°F; July 1987.

Problem & Solution: "Similar to case 3 above in symptoms, and again, same excellent results: Steel Shield EPA added and system operating normally."



FIGURE 5 - COPELAND KAJ-0100 COMPRESSOR



FIGURE 6 - TECUMSEH 3/4 HP COMPRESSOR

CASE 5

System Configurations: 42" Belt-drive exhaust fan, propeller type; 1" pillow block bearings.

Problem & Solution: "Because of constant start-stop cycling, and also because of just marginal design, fanshaft bearings were a constant replacement item, requiring replacement about every 12-15 months. A new set of the same type were installed, then lubricated with Steel Shield - Lithi Shield. Same

CASE 6

Problem & Solution: "Countless sleeve-bearing fan motors, many locked or very near locked; Steel Shield EPA mixed with 150 vis. refrigeration oil in about a 50/50 ratio, added directly to bearings. At least 75% of all motors treated responded. The customer used this mix in the routine maintenance of

"We could go on (and on!), but this should give you a little ideo of how successful Steel Shield has been for the customers in the HVAC service. We have several customers who insist that we add Steel Shield EPA to any equipment they have or add. Almost without exception, they will add and said that they didn't care what Steel Shield costs, just make sure we add it to their unit!"

FIGURE 7 - PILLOW BLOCK BEARING



FIGURE 8 - SLEEVE BEARINGS



1993 - MURPHY HEATING & AIR CONDITIONING

On February, 1990, A TRANE 71/2 ton rooftop unit, Model No. YCH090A3L0AA, Serial No. D29145142D was installed in Pawn-Pub Cocktail Lounge, 1000 Village Green, Universal City, TX. The air-conditioning equipment was running 16 hours per day.

8/5/93. On 8/23/93, the energy consumption of the system situation of 22.5 amps. Energy consumption saved over 16%.

Date	Amperage Compressor	Suction Pressure	Liquid Pressure	Ambient Temperature	Indoor Temperature
5/31/93	22.5	68	285	91.1	77
5/31/93	Added 6 oz. Steel Shield EPA				
6/30/93	21.5	61	260	90	69.3
8/05/93	21.8	70	275	92	
8/05/93	Added 6.5 oz. Steel Shield EPA				
8/23/93	18.8	63	280	96	75



Table 4- OPERATING CONDITIONS OF TRANE

EQUIPMENT On 8/23/93, after Steel Shield EPA was added, current dropped dramatically to only 18.8 amps

2003 - KOWLOON CITY PLAZA

The Kowloon City Plaza have 8 units of Carrier reciprocate chillers, 4 of them are using Steel Shield lubricants. All of the compressors have individual ampere meter to record energy consumption activities. From August, 2003 to January, 2004, the systems operated 12 hours per day.

The engineers of Kowloon City Plaza concluded that Steel Shield dramatically reduced energy consumptions from 5% to 18% which was much more economic than Carrier lubricants. significantly.



FIGURE 9 - CARRIER RECIPROCATE CHILLER SYSTEM





22. MAJOR CORPORATE CLIENTS

"Your trust is our motivation."



22A. THE MOST REPUTED CUSTOMERS



UNITED STATES • The most advanced army in the world

SIEMENS

SIEMENS

• The largest engineering company in Europe

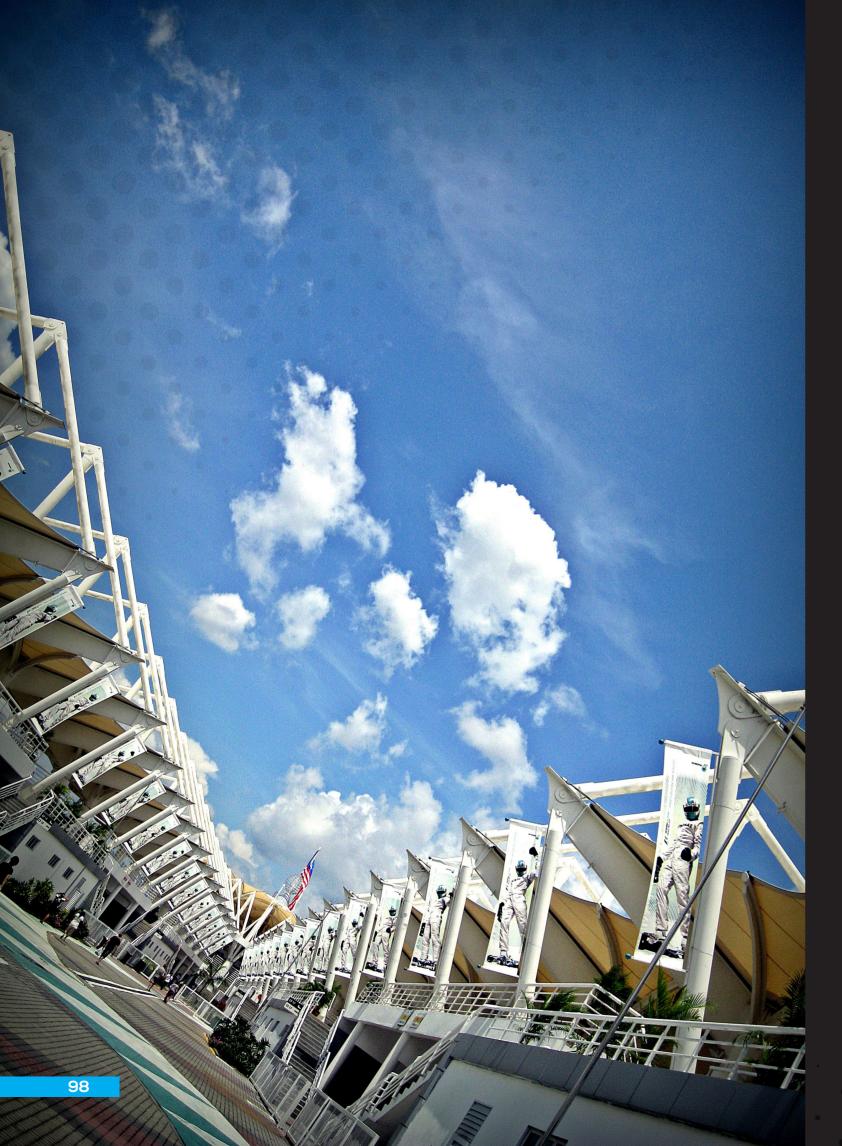


UNION PACIFIC RAILROAD • The largest railway company in the US • New York Stock Exchange No.: UNP



• A leading provider for wastes

- A leading provider for wastes management and environmental services in the PRC
 Hong Kong Stock Exchange No.: 895



23. STEEL SHIELD EVENTS

"Practical experience with STEEL SHIELD lubricants."



61ST MACAU GRAND PRIX 2014









ZHUHAI 3 HOURS MOTORCYCLE COMPETITION 2014



60[™] MACAU GRAND PRIX 2013





USA RACING EVENTS







23B. STEEL SHIELD REMARKABLE EXHIBITIONS

HONG KONG MOTORCYCLE SHOW 2014









HONG KONG MOTORCYCLE SHOW 2013







QINGDAO LUBRICANT SHOW 2013











24. STEEL SHIELD OFFICES & FACILITIES

"Consolidated manufacturing plant of ABF Technology and management offices enabled STEEL SHIELD to expand the service networks to differenct sectors all over the world."



24A. STEEL SHIELD TECHNOLOGIES US HEADQUARTER

24B. STEEL SHIELD TECHNOLOGIES H.K. OFFICE

THE BASE OF ABF TECHNOLOGIES



HEADQUARTER OF ASIA PACIFIC NETWORK









25. STEEL SHIELD WEBSITE & VIDEOS

"Up-to-Date information and demonstrations are available through mouse clicks."

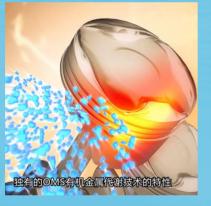


STEEL SHIELD VIDEOS DESCRIPTIONS



STEEL SHIELD TECHNOLOGY INTRODUCTION

How ABF Technology works?
What benefits can you gain from Steel Shield?



STEEL SHIELD TECHNOLOGY **INTRODUCTION (CHINESE)**

- How ABF Technology works?
 What benefits can you gain from Steel Shield?
 Chinese dialogue and subtitles



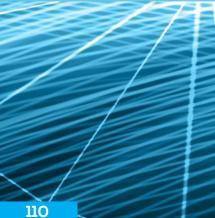
TIMKEN TEST DEMONSTRATION

- How Steel Shield lubricants defeat other preminum lubricants in the market?
 How ABF Technology dramatically reduce metal wear, maintenance costs and energy?



GUANGDONG SPORT TV INTERVIEW Steel Shield being recongized by the Chinese customers and market

STEEL SHIELD MOTORHEAD **GARAGE COMMERCIAL** The original US promotion show introducing Steel Shield products











STEEL SHIELD TECH FULL FEATURE ON MOTORHEAD GARAGE

The original US video showing how Steel Shield lubricants with ABF Technology enhance engine performance



STEEL SHIELD TECHNOLOGIES

www.steelshieldtech.com.hk

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